



Harley-Davidson Museum

Milwaukee, WI.

Jonathan Rumbaugh, BAE/MAE
Mechanical Option

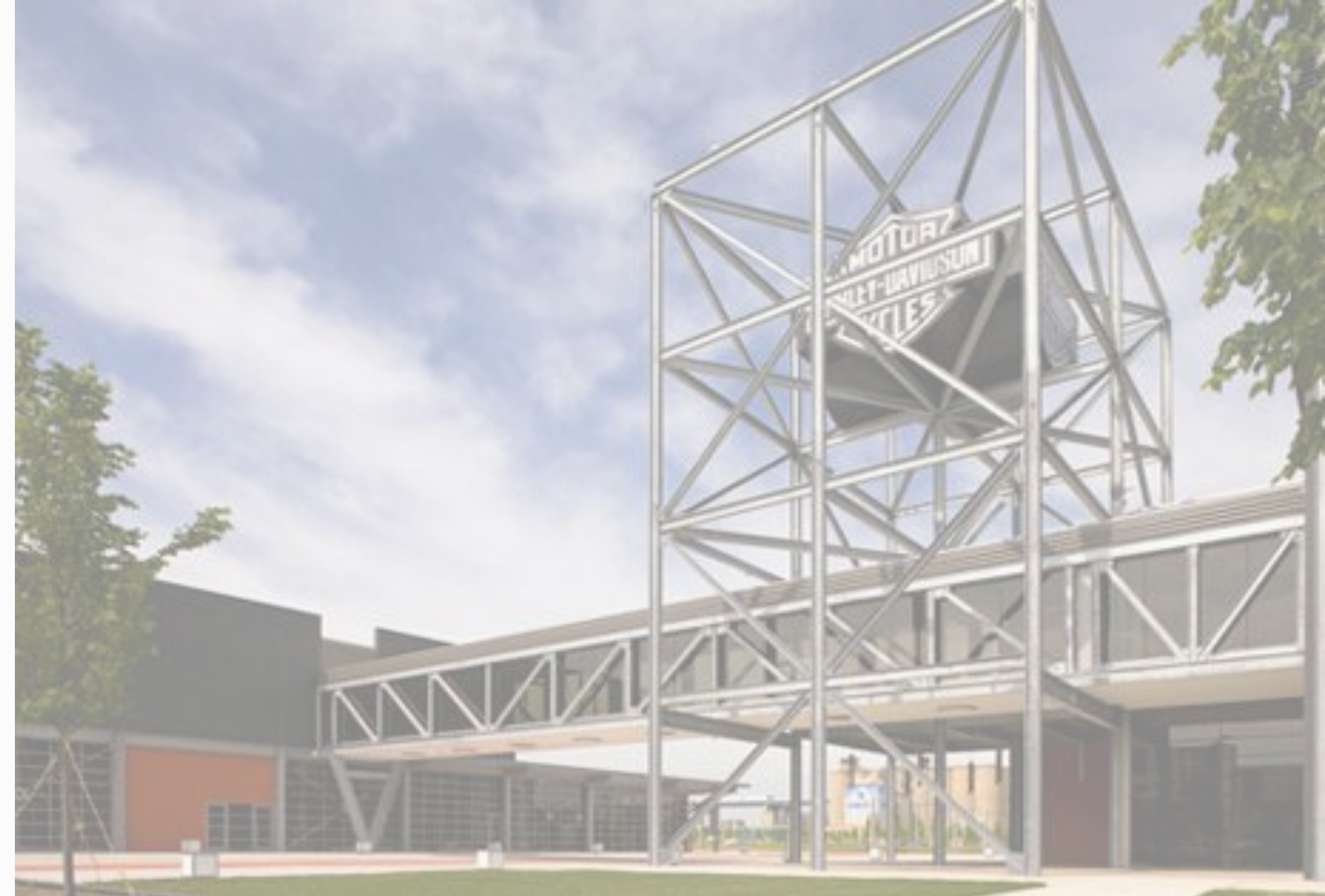
Advisor: Dr. William Bahnfleth



PRESENTATION OUTLINE

- Project Background
- Existing Conditions
- Thesis Goals
- Structural Breadth:
 - Thermal Bridging
- Mechanical Depth:
 - Air vs. Water
- Electrical Breadth
 - CHP Feasibility
- Conclusion

PROJECT SPONSORS



PROJECT BACKGROUND

- **Project Background**
 - **Building Statistics**
 - **Location & Layout**
- Existing Conditions
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion

BUILDING STATISTICS

- Size (Total Square Feet): 130,000
- Number Stories Above grade: 3
- Construction timeline : April 2005 – May 2008
- Overall Project Cost: \$75 million



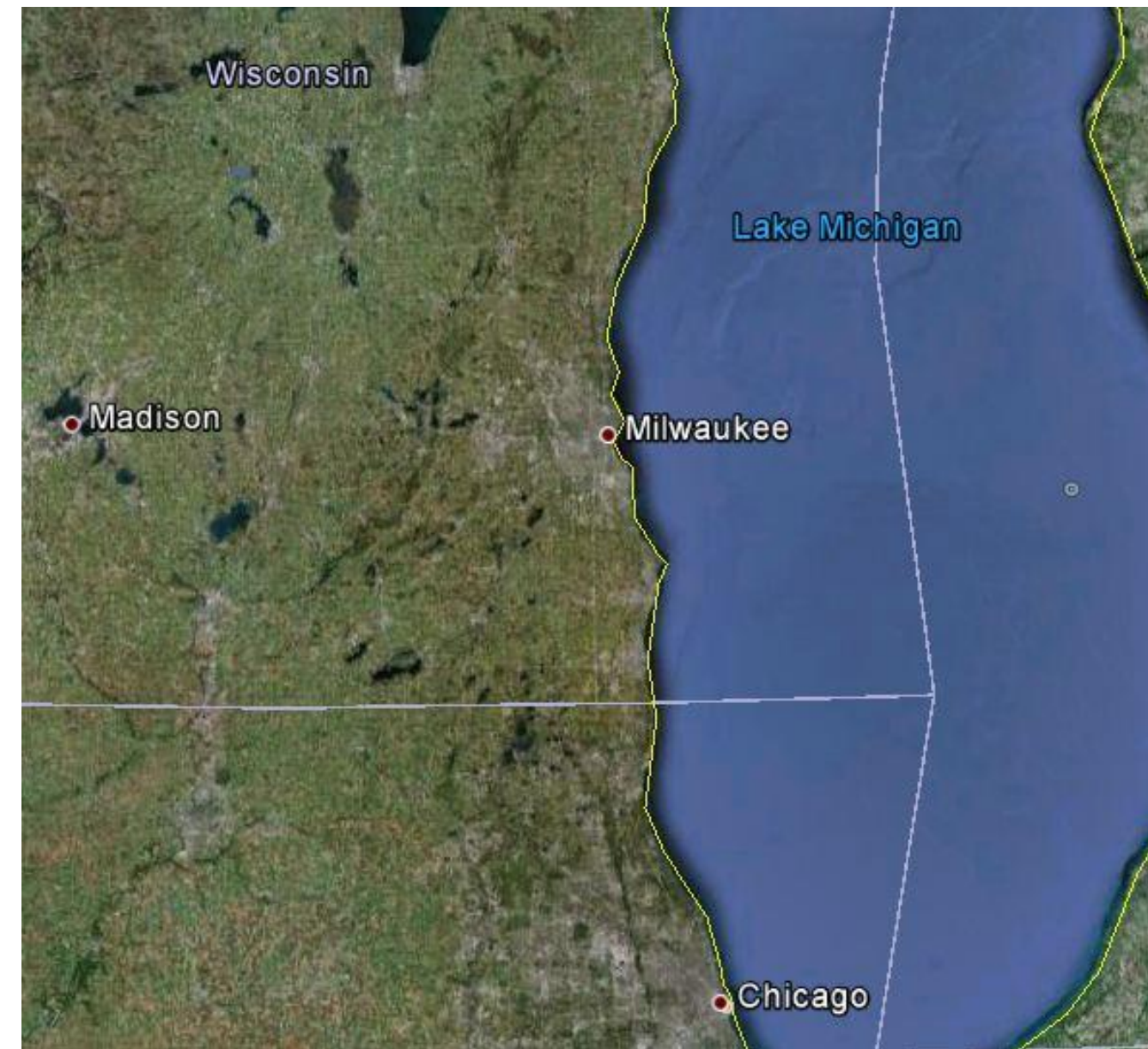
Guggenheim by Frank Gehry
257,000 square feet
\$100 million



Wordpress.com

PROJECT BACKGROUND

- **Project Background**
 - Building Statistics
 - **Location & Layout**
- Existing Conditions
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion



EXISTING CONDITIONS

- Project Background
- Existing Conditions
 - Mechanical Design
 - Energy Model
 - Emissions
 - Comparison
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion

MECHANICAL DESIGN

- Two Roof Mounted 300 Ton Air-cooled Rotary Screw Chillers
- Four 2000 MBH Sealed Combustion Condensing Boilers
- Variable Primary Flow
- 11 Air Handling Units



EXISTING CONDITIONS

ENERGY MODEL

- Project Background
- Existing Conditions
 - Mechanical Design
 - Energy Model
 - Emissions
 - Comparison
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion



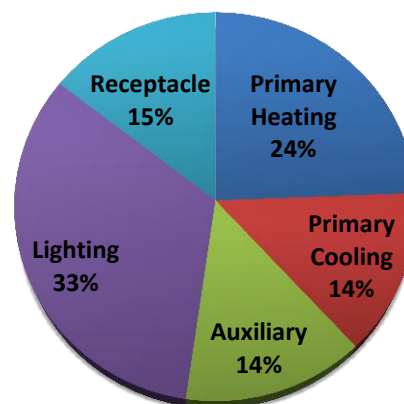
Peak Cooling Plant Loads		
Desgin	TRACE MODEL	Design to Model
ton	ton	%Δ
600	585.3	-2%

Peak Heating Plant Loads		
Desgin	TRACE MODEL	Design to Model
MBh	MBh	%Δ
8000	9073	13%

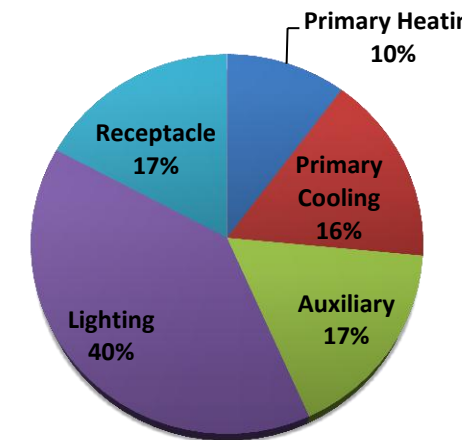
\$0.10 / kWh Electricity
\$0.80 / Therm Natural Gas



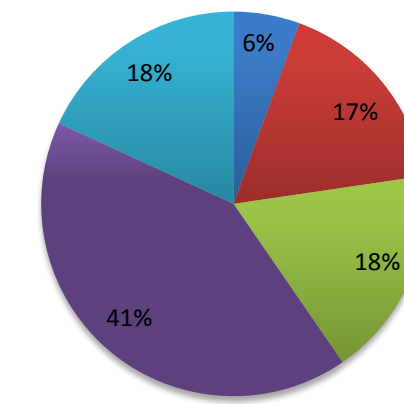
Total Building Energy [kBtu/yr]



Total Source Energy [kBtu/yr]



Annual Cost Breakdown



- Primary Heating
- Primary Cooling
- Auxiliary
- Lighting
- Receptacle

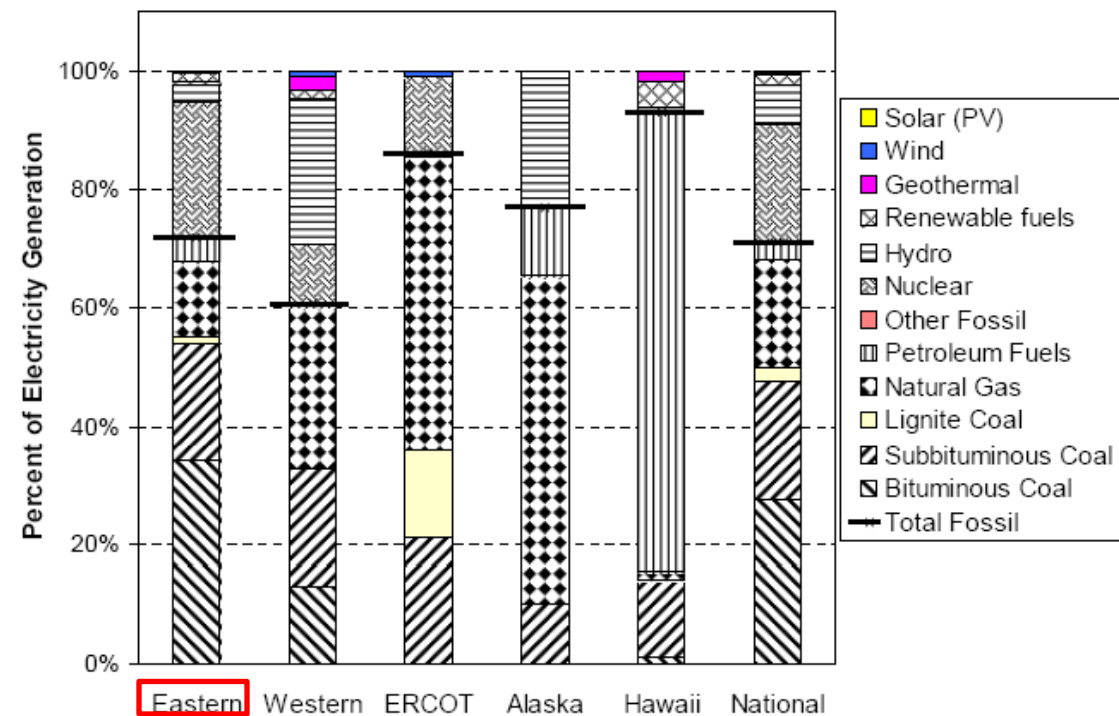
Cost Breakdown	
	Cost
Primary Heating	\$ 20,252.00
Primary Cooling	\$ 62,223.50
Auxiliary	\$ 64,463.40
Lighting	\$ 150,907.60
Receptacle	\$ 65,906.60
Total	\$ 363,753.10

EXISTING CONDITIONS

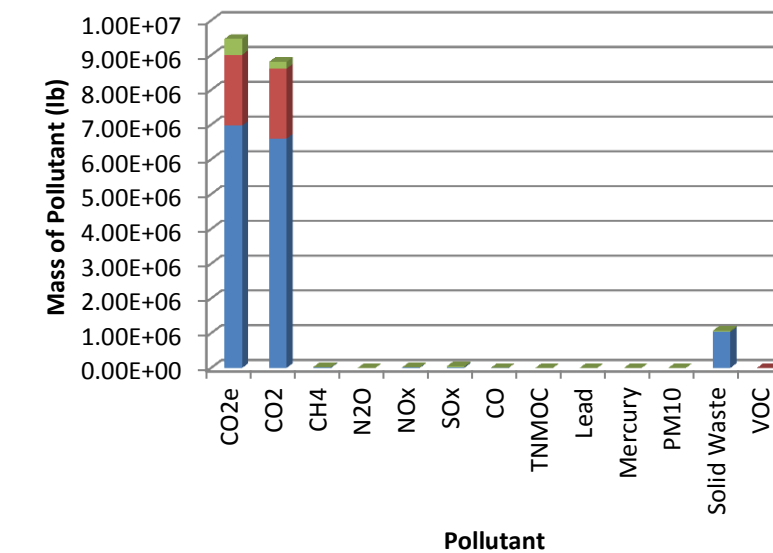
- Project Background
- Existing Conditions
 - Mechanical Design
 - Energy Model
 - Emissions
 - Comparison
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion



EMISSIONS

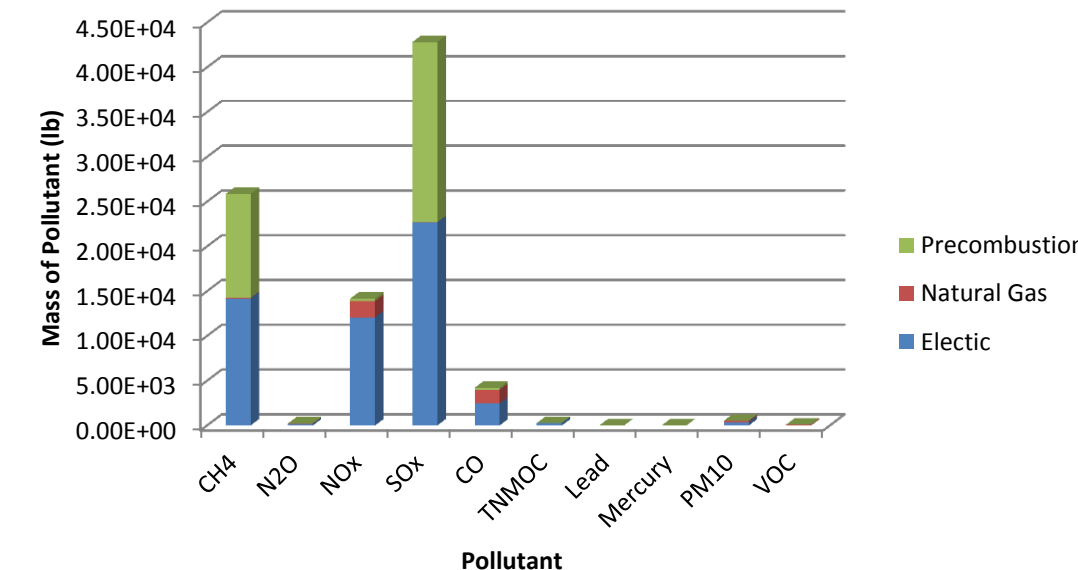


Emissions



Calculations use factors from the Regional Grid emissions Factors 2007. table B-10

Emissions : No CO₂



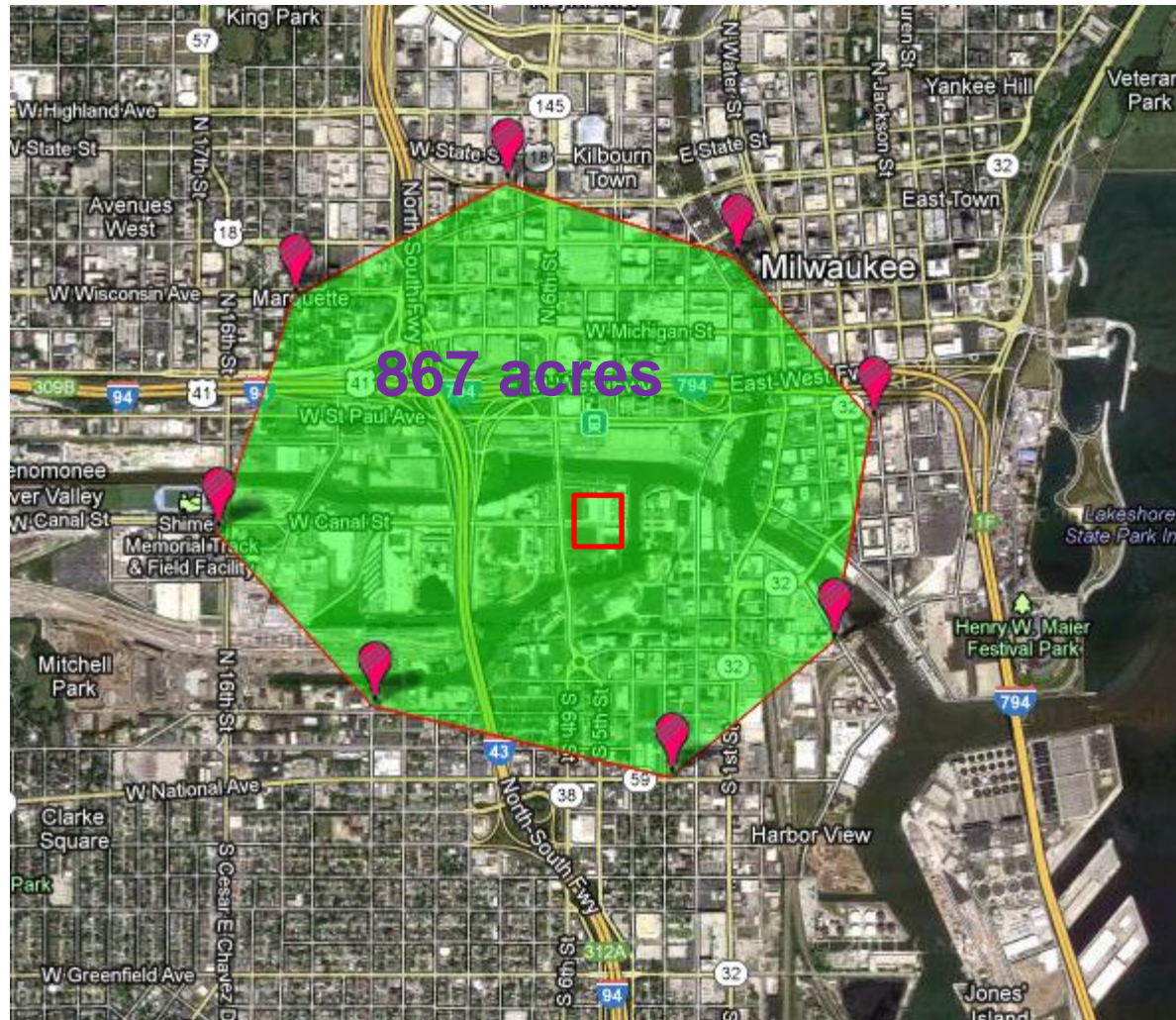
EXISTING CONDITIONS

- Project Background
- Existing Conditions
 - Mechanical Design
 - Energy Model
 - Emissions
 - Comparison
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion

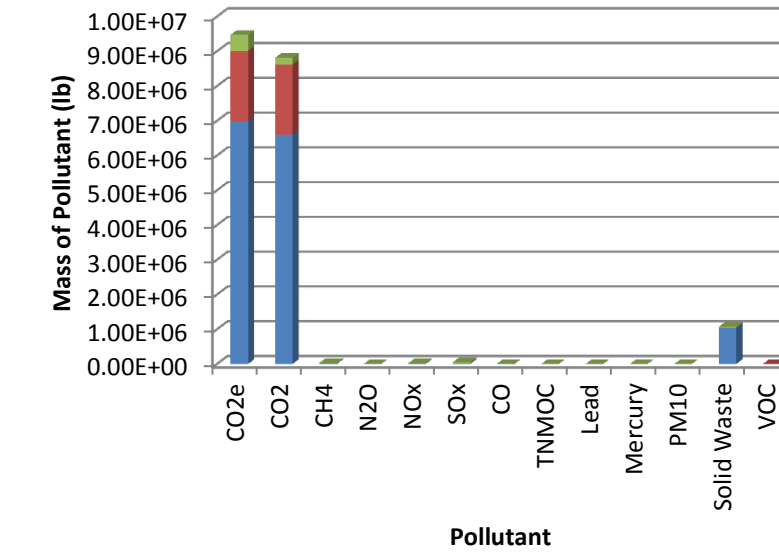
9 Million lbs. of CO_{2e} / year



EMISSIONS

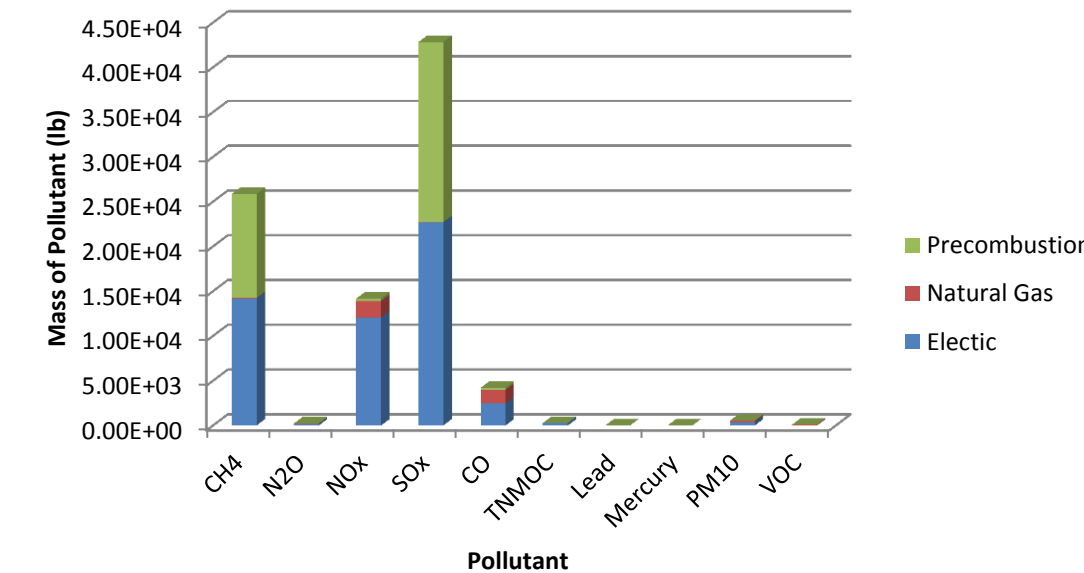


Emissions



Calculations use factors from the Regional Grid emissions Factors 2007. table B-10

Emissions : No CO₂



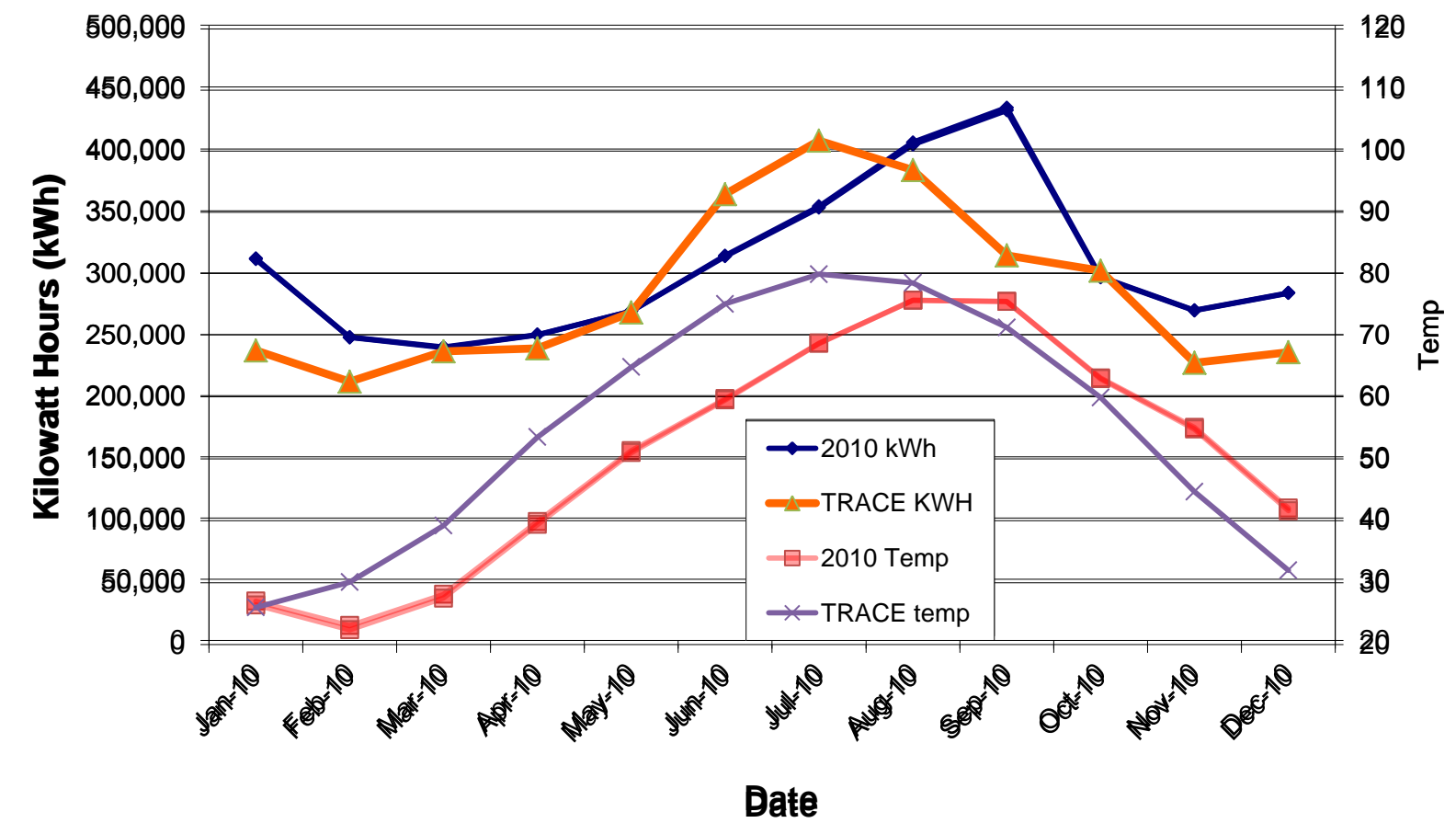


EXISTING CONDITIONS

- Project Background
- Existing Conditions
 - Mechanical Design
 - Energy Model
 - Emissions
 - Comparison
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion

COMPARISON

Museum Campus Electricity Use



CONCLUSION

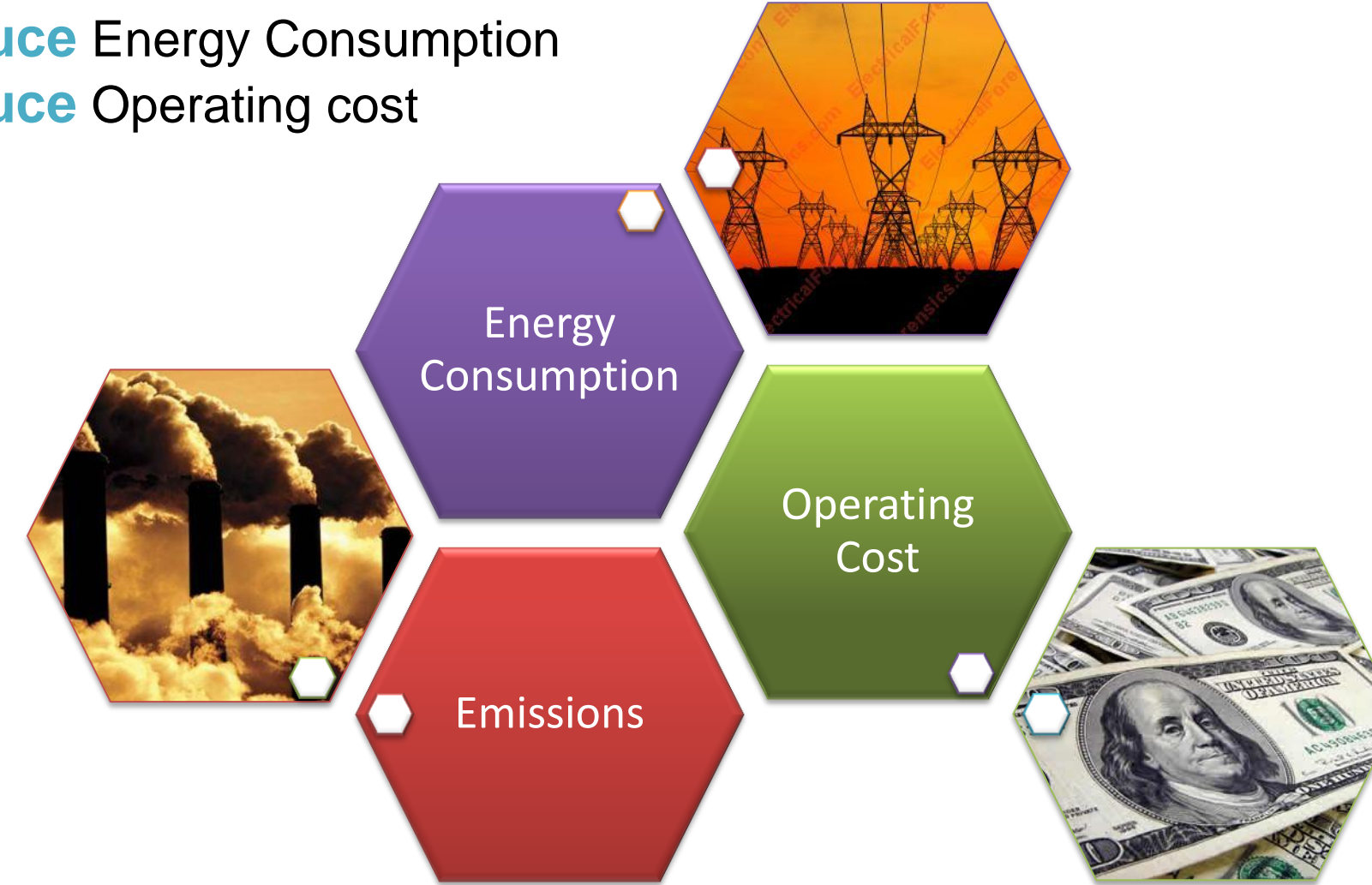
Energy Model is an accurate representation of the energy consumption of the facility

THESIS GOALS

- Project Background
- Existing Conditions
- **Thesis Goals**
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- Conclusion

GOALS

- **Reduce** Emissions
- **Reduce** Energy Consumption
- **Reduce** Operating cost



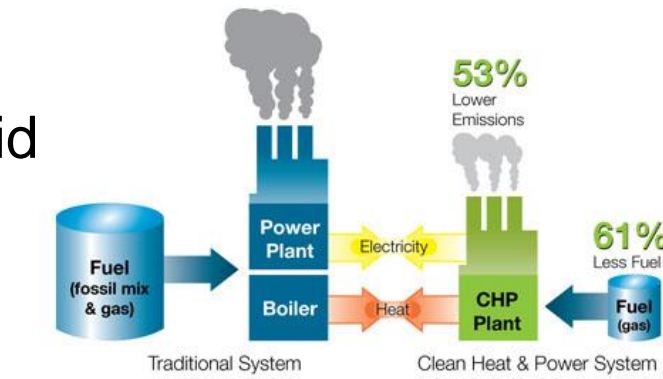
PROPOSAL

- Decrease Thermal Loads Through Envelope



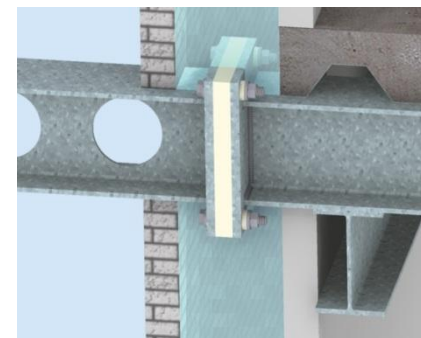
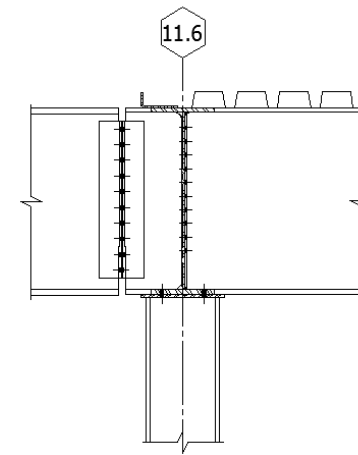
- Increase Efficiency of Chilled Water Production

- Become Energy Independent From Grid

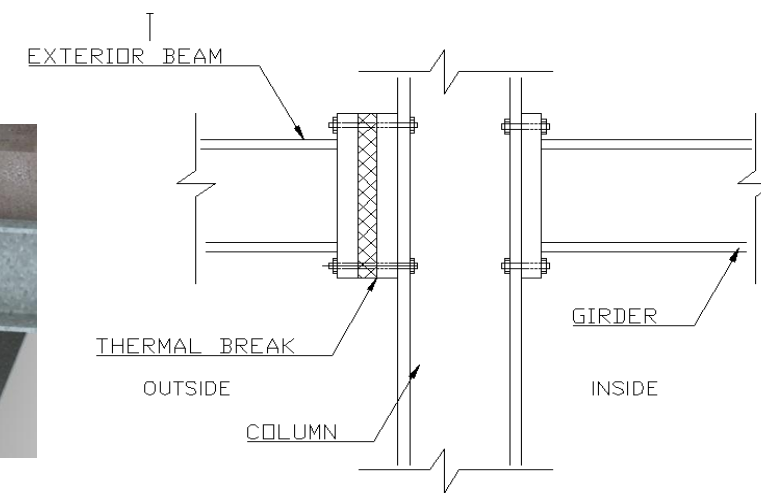


STRUCTURAL BREADTH

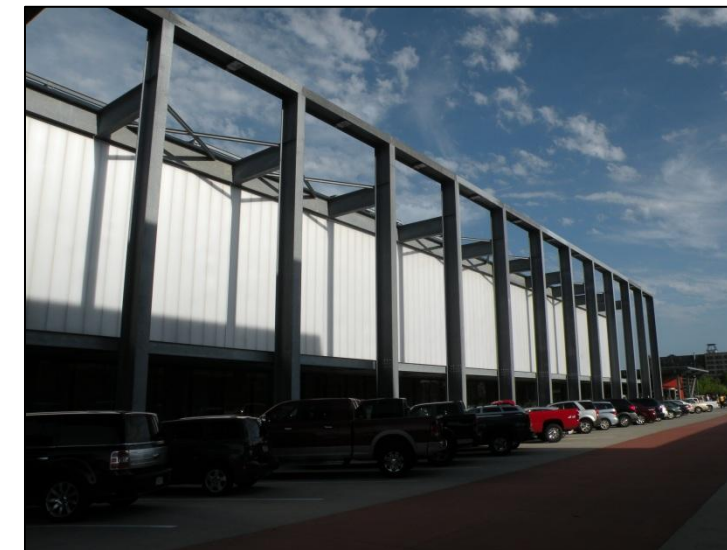
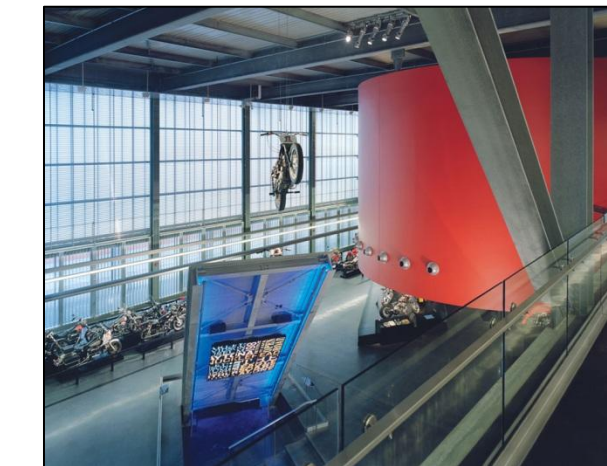
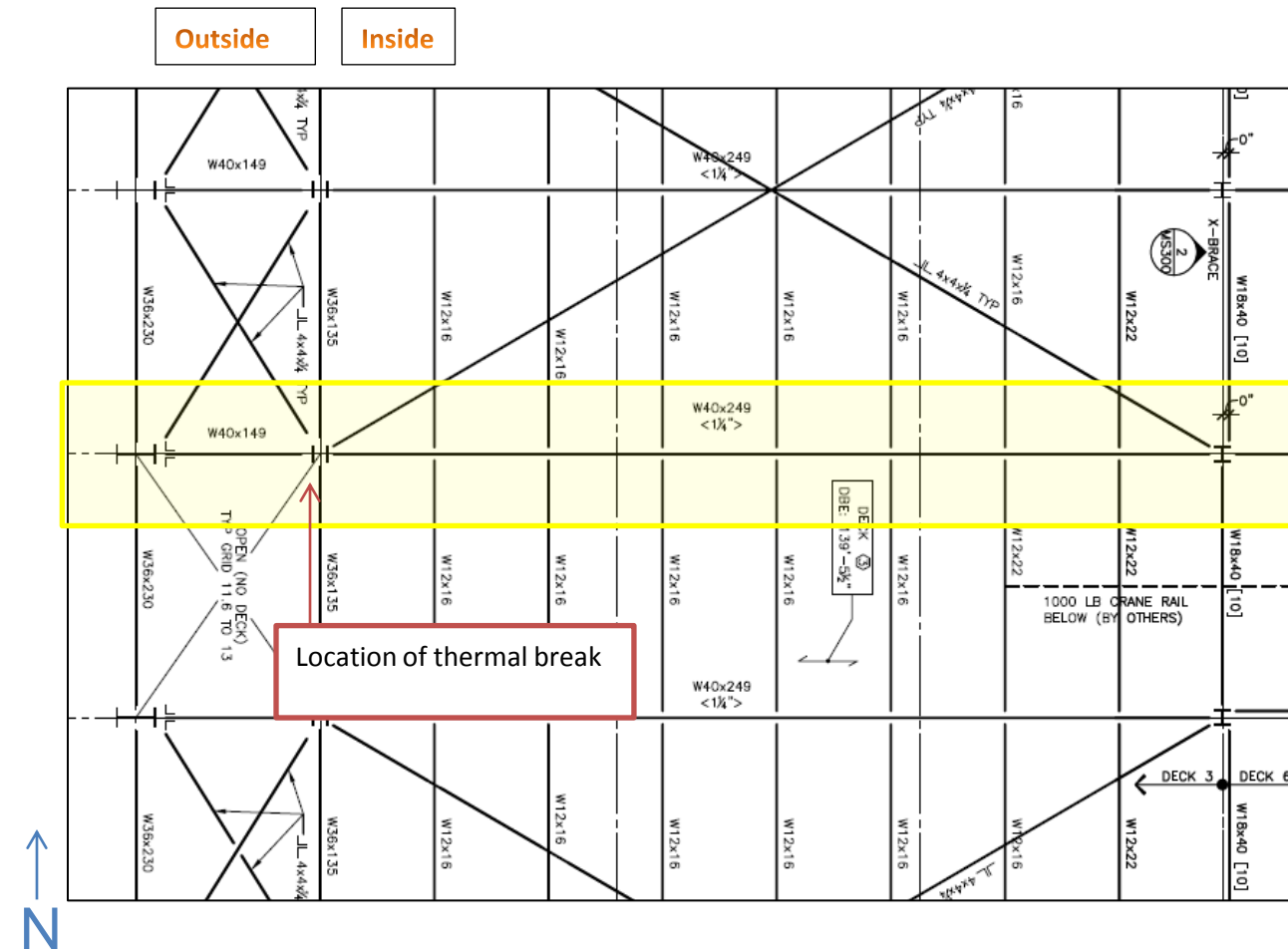
- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - Thermal Analysis
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion



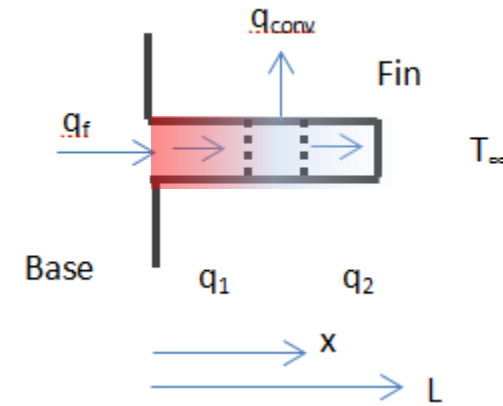
FABREEKA



THERMAL BRIDGING



- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - **Thermal Analysis**
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion



Starting with the conservation of energy equation

$$q_2 = q_1 + dq \text{ (From Conservation of Energy)}$$

$$q_x = -kA_c \frac{dT}{dx} \text{ (From Fourier's Law)}$$

$A_c = \text{Cross section area}$, $k = \text{conductivity of the material}$

$$dq_{conv} = h dA_s (T - T_\infty)$$

$$\frac{d}{dx} \left[-kA_c \frac{dT}{dx} \right] dx = hA_s (T - T_\infty)$$

$$\frac{d^2T}{dx^2} - \frac{hP}{kA_c} (T - T_\infty) = 0$$

$$\frac{d^2\theta}{dx^2} - m^2\theta = 0 \text{ (Homogeneous Second Order ODE)}$$

$$m^2 \equiv \left(\frac{hP}{kA_c} \right), \theta = T - T_\infty$$

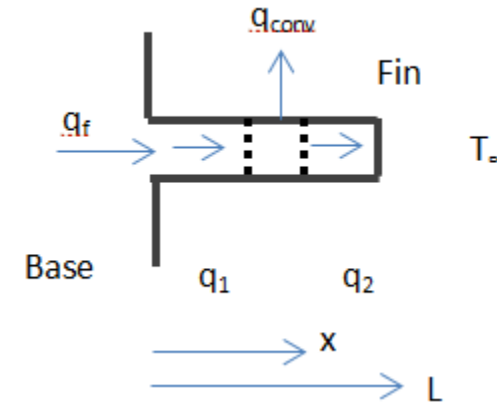
$$\frac{d^2\theta}{dx^2} - m^2\theta = 0 \rightarrow \theta(x) = C_1 e^{mx} + C_2 e^{-mx} \text{ (General Solution)}$$

Boundary Conditions:

1. Adiabatic Tip: $\left. \frac{dT}{dx} \right|_{x=L} = 0$
2. One Direction Heat Transfer: $\Delta T(\text{base}) = T(\text{base}) - T_\infty$

STRUCTURAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - **Thermal Analysis**
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion



THERMAL ANALYSIS

$$C_1 = \frac{\theta_b e^{-mL}}{e^{mL} + e^{-mL}} \quad C_2 = \theta_b - C_1$$

- Conduction at the base is equal to the total convective heat transfer

$$q_f = -kA_c \left. \frac{d\theta}{dx} \right|_{x=0} = \int_{A_F} h\theta(x) dA_s \quad \left. \vphantom{q_f} \right\} q_{cond} \text{ at base of fin}$$

$$q_f = -kA_c \left. \frac{dT}{dx} \right|_{x=0} = M \tanh(mL)$$

$$m = \sqrt{\frac{hP}{kA_c}}, \quad M = \sqrt{hPkA_c} (T_b - T_\infty), \quad L = \text{length of fin}$$

$$\frac{d^2 T}{dx^2} - \frac{hP}{kA_c} (T - T_\infty) = 0$$

$$\frac{d^2 \theta}{dx^2} - m^2 \theta = 0 \quad (\text{Homogeneous Second Order ODE})$$

$$m^2 \equiv \left(\frac{hP}{kA_c} \right), \quad \theta = T - T_\infty$$

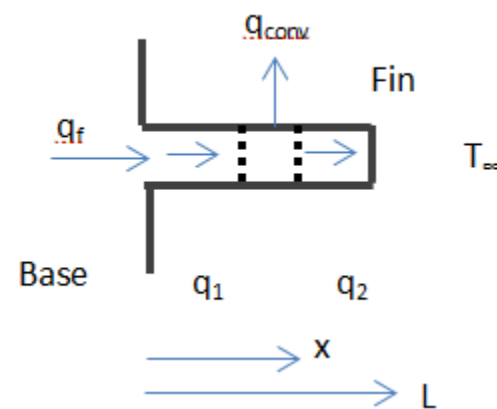
$$\frac{d^2 \theta}{dx^2} - m^2 \theta = 0 \rightarrow \theta(x) = C_1 e^{mx} + C_2 e^{-mx} \quad (\text{General Solution})$$

Boundary Conditions:

1. Adiabatic Tip: $\left. \frac{dT}{dx} \right|_{x=L} = 0$
2. $\Delta T(\text{base}) = T(\text{base}) - T_\infty$

STRUCTURAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - **Thermal Analysis**
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion



THERMAL ANALYSIS

$$C_1 = \frac{\theta_b e^{-mL}}{e^{mL} + e^{-mL}} \quad C_2 = \theta_b - C_1$$

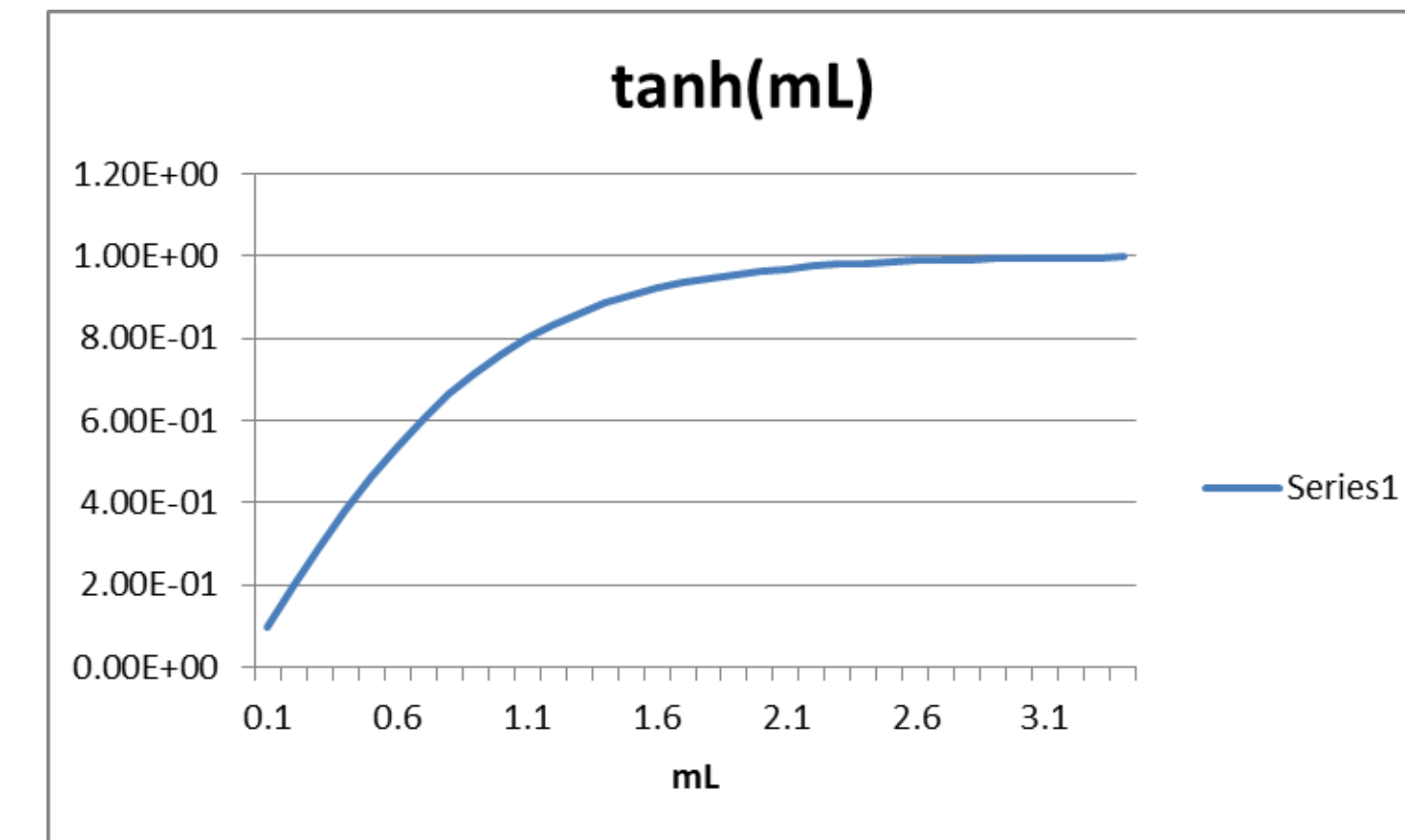
- Conduction at the base is equal to the total convective heat transfer

$$q_f = -kA_c \left. \frac{d\theta}{dx} \right|_{x=0} = \int_{A_F} h\theta(x) dA_s \quad \left. \vphantom{q_f} \right\} q_{cond} \text{ at base of fin}$$

$$q_f = -kA_c \left. \frac{dT}{dx} \right|_{x=0} = M \tanh(mL)$$

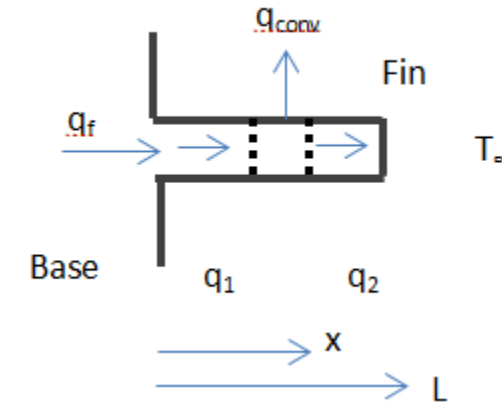
$$m = \sqrt{\frac{hP}{kA_c}}, \quad M = \sqrt{hPkA_c} (T_b - T_\infty), \quad L = \text{length of fin}$$

$$q_f = -kA_c \left. \frac{dT}{dx} \right|_{x=0} = \sqrt{hPkA_c} (T_b - T_\infty) \tanh(mL)$$

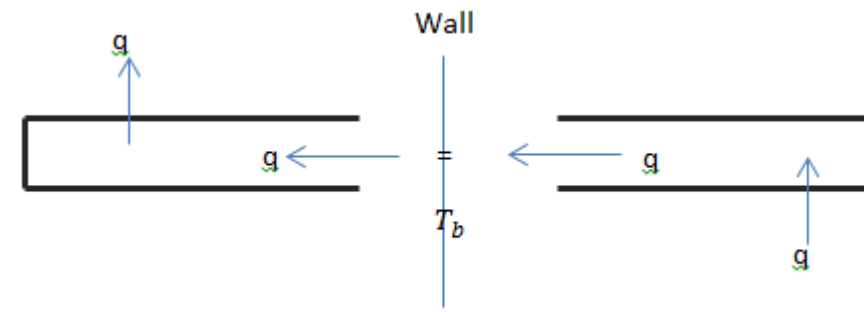


STRUCTURAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - **Thermal Analysis**
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion



THERMAL ANALYSIS



$$q_{outside} = -q_{inside} \rightarrow M_{outside} = M_{inside}$$

$$\sqrt{h_o P k A_c} (T_b - T_{\infty_o}) = -\sqrt{h_i P k A_c} (T_b - T_{\infty_i})$$

$$T_b = \frac{\sqrt{h_i} T_{\infty_i} + \sqrt{h_o} T_{\infty_o}}{\sqrt{h_o} + \sqrt{h_i}} = \mathbf{29.8^\circ F}$$

$$q = M = \sqrt{h P k A_c} (T_b - T_{\infty}) = \mathbf{422 \text{ Watts}}$$

Constants:

$$k = \text{Conductivity of steel} = 43 \frac{W}{m^2 K}$$

$$A_c = \text{Cross section area of a W40x149} = 0.047 \text{ m}^2$$

$$h_o = \text{Convective heat transfer coefficient of outside air} = 30 \frac{W}{m^2 K}$$

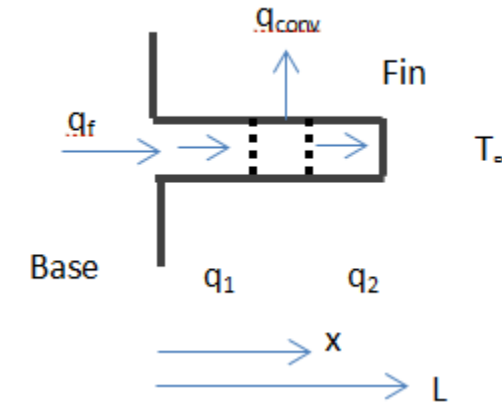
$$h_i = \text{Convective heat transfer coefficient of inside air} = 15 \frac{W}{m^2 K}$$

$$T_{\infty_o} = \text{Outside air temperature} = 0^\circ F$$

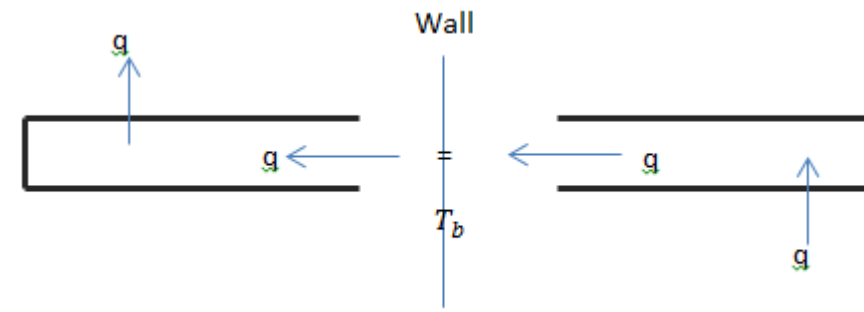
$$T_{\infty_i} = \text{Inside air temperature} = 72^\circ F$$

STRUCTURAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - **Thermal Analysis**
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion



THERMAL ANALYSIS



$$q_{outside} = -q_{inside} \rightarrow M_{outside} = M_{inside}$$

$$\sqrt{h_o P k A_c} (T_b - T_{\infty_o}) = -\sqrt{h_i P k A_c} (T_b - T_{\infty_i})$$

$$T_b = \frac{\sqrt{h_i} T_{\infty_i} + \sqrt{h_o} T_{\infty_o}}{\sqrt{h_o} + \sqrt{h_i}} = 29.8^\circ F < 53^\circ F \text{ dp}$$

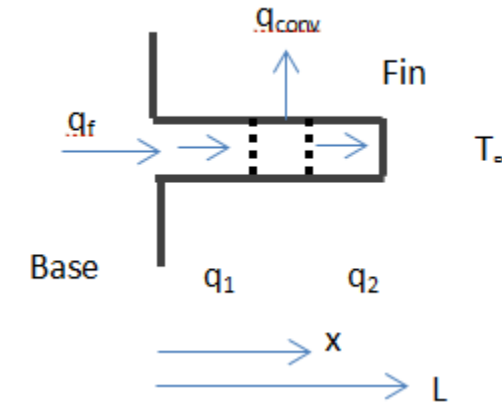
$$q = M = \sqrt{h P k A_c} (T_b - T_{\infty}) = 422 \text{ Watts}$$

8760 hr. study

Total: 272,407.12 Watts per year

STRUCTURAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - **Thermal Analysis**
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion



THERMAL ANALYSIS

8760 hr. study

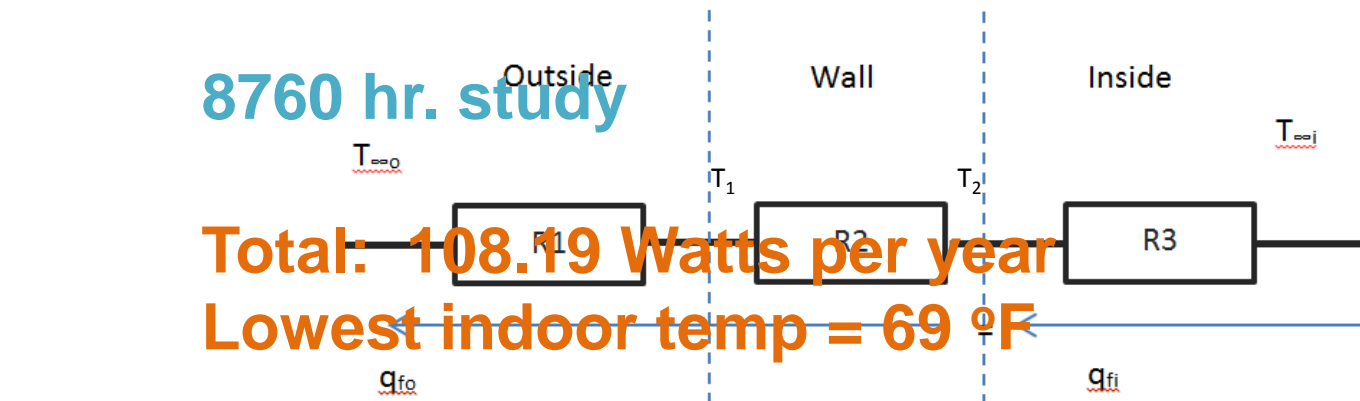
Total: 108.19 Watts per year

Lowest indoor temp = 69 °F

Savings: \$1,271.19 / year [Main gallery]

For a simple payback of 5 years

Each Thermal Break = \$653



$$R1 = \frac{\Delta T}{q} \rightarrow \frac{\Delta T}{\sqrt{h_o P k A_c} \Delta T} \rightarrow \frac{1}{\sqrt{h_o P k A_c}}$$

$$R2 = \frac{\Delta T}{k_r A_c} \text{ Thermal Break}$$

$$R3 = \frac{\Delta T}{q_{fi}} \rightarrow \frac{\Delta T}{\sqrt{h_i P k A_c} \Delta T} \rightarrow \frac{1}{\sqrt{h_i P k A_c}}$$

$$T_2 = \frac{\sqrt{h_i P k A_c} \left(\frac{1}{\sqrt{h_o P k A_c}} + \frac{L_r}{k_r A_c} \right) T_{\infty_o} + T_{\infty_i}}{1 + \sqrt{h_i P k A_c} \left(\frac{1}{\sqrt{h_o P k A_c}} + \frac{L_r}{k_r A_c} \right)}$$

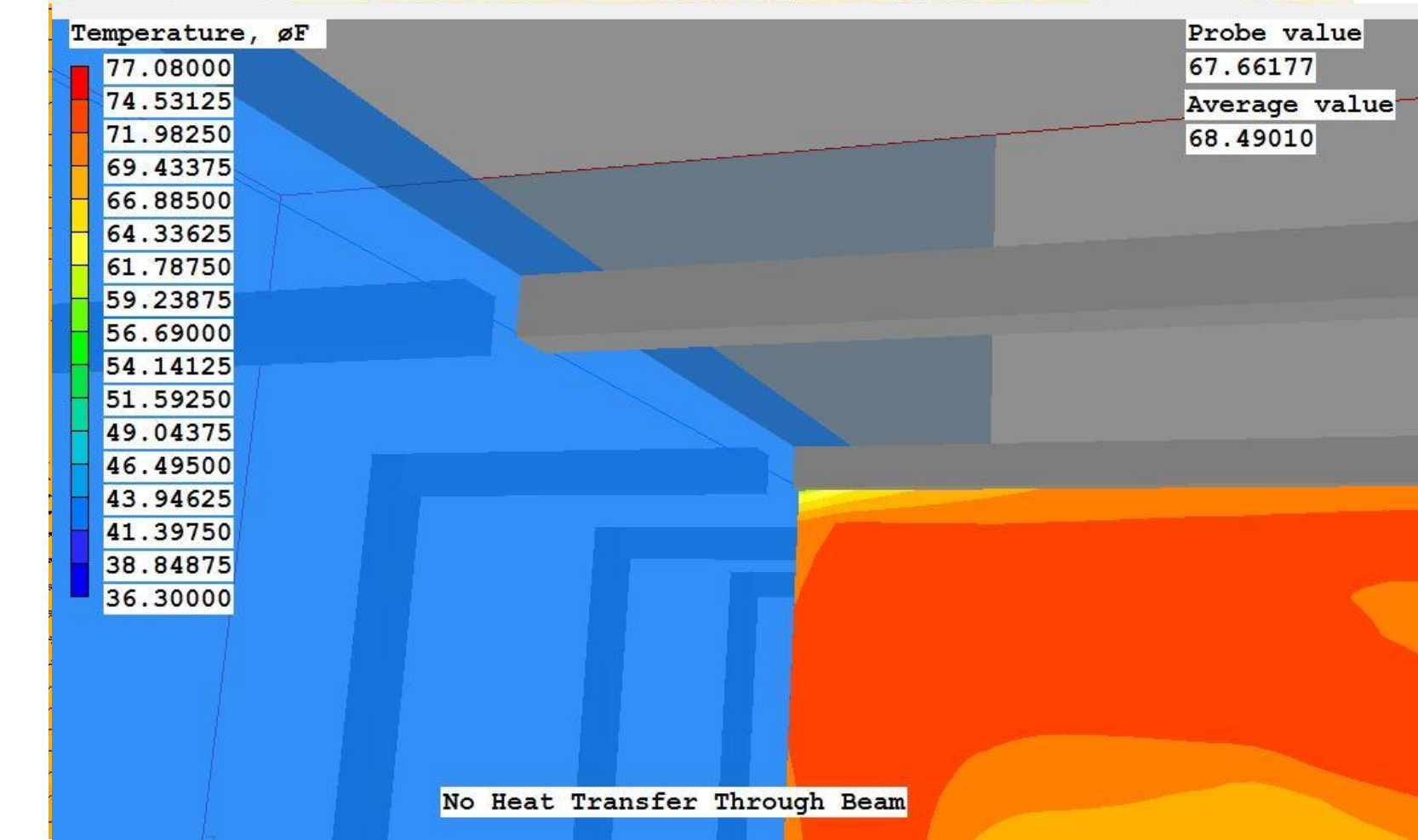
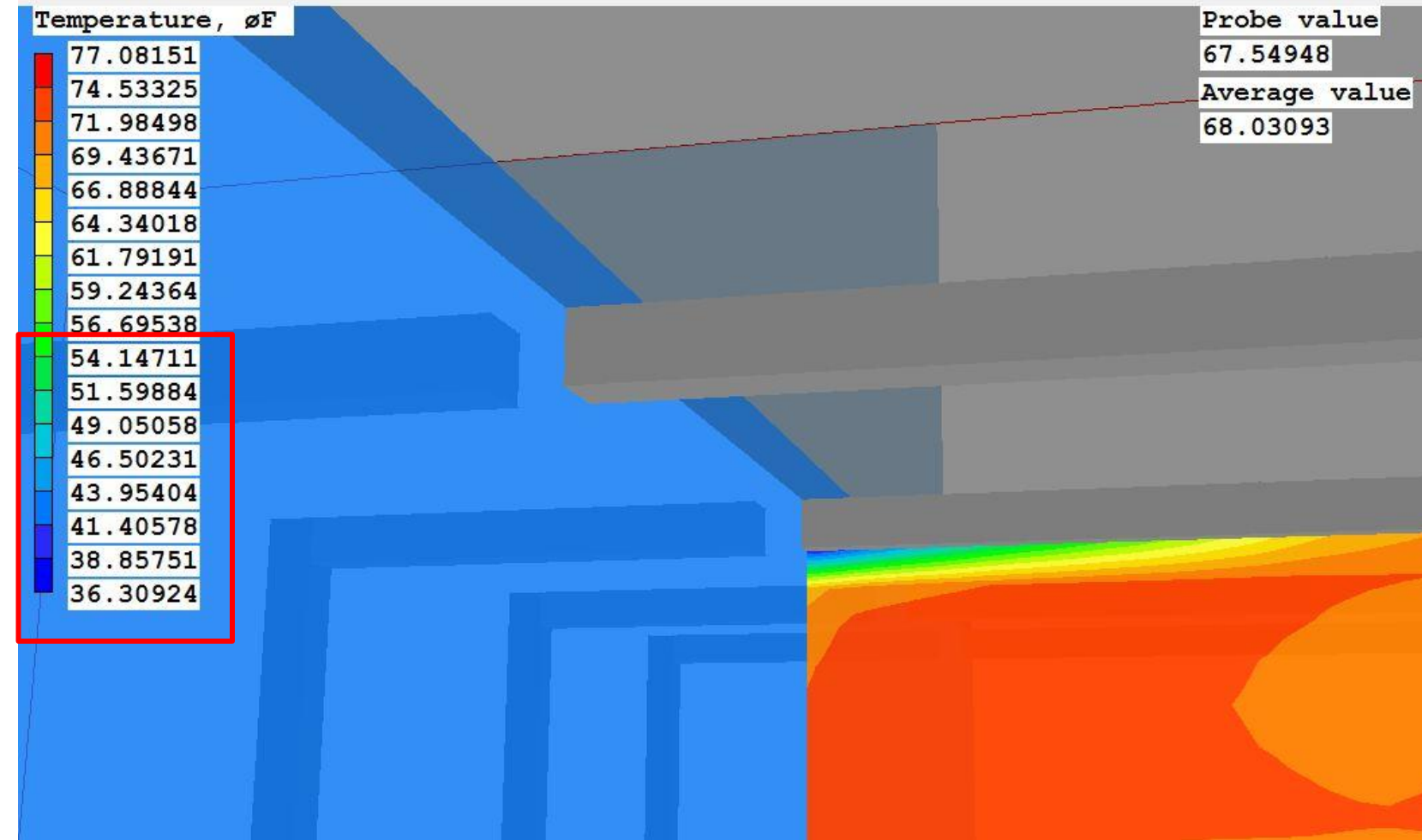
$$q = -\sqrt{h_i P k A_c} (T_2 - T_{\infty_i})$$

STRUCTURAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - **Thermal Analysis**
 - Structural Analysis
- Mechanical Depth
- Electrical Breadth
- Conclusion

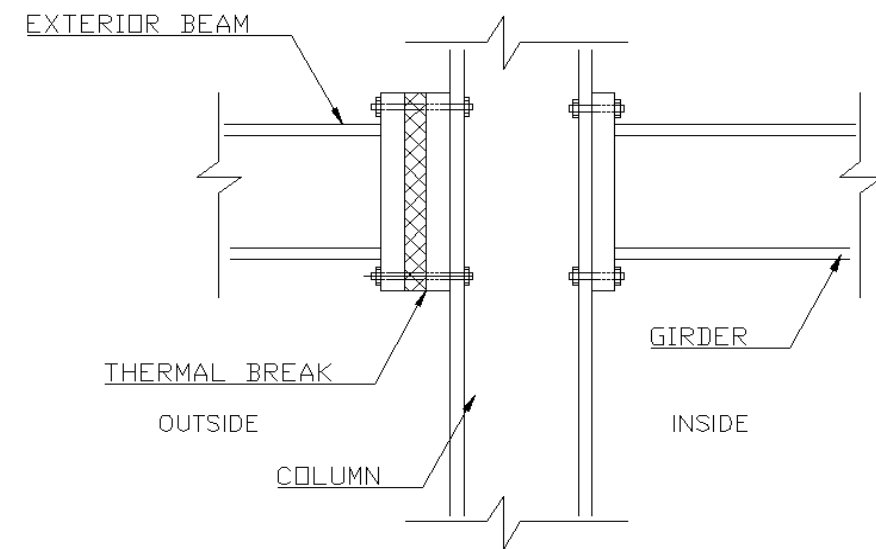


THERMAL ANALYSIS

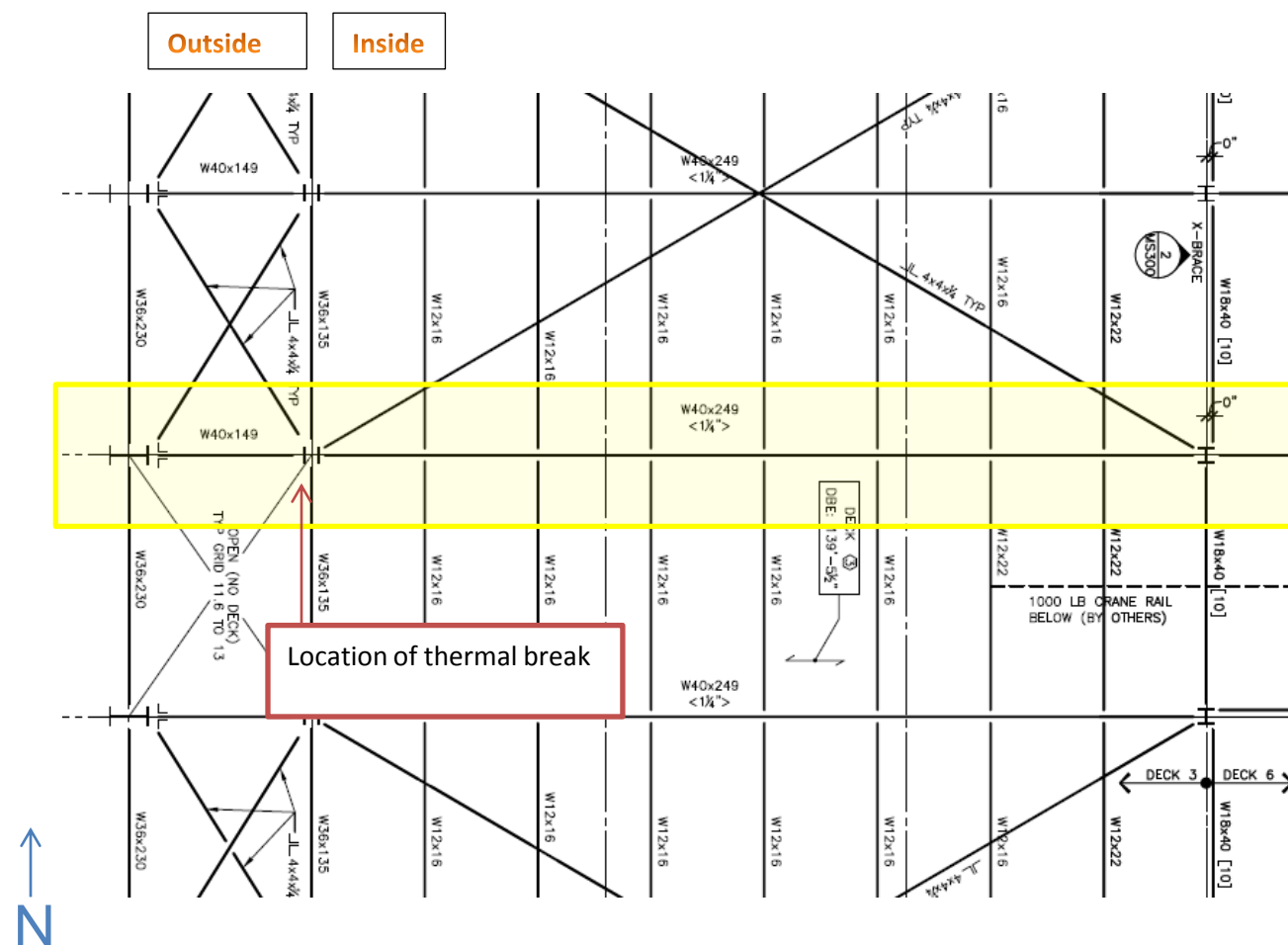


STRUCTURAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- **Structural Breadth**
 - Thermal Analysis
 - **Structural Analysis**
- Mechanical Depth
- Electrical Breadth
- Conclusion



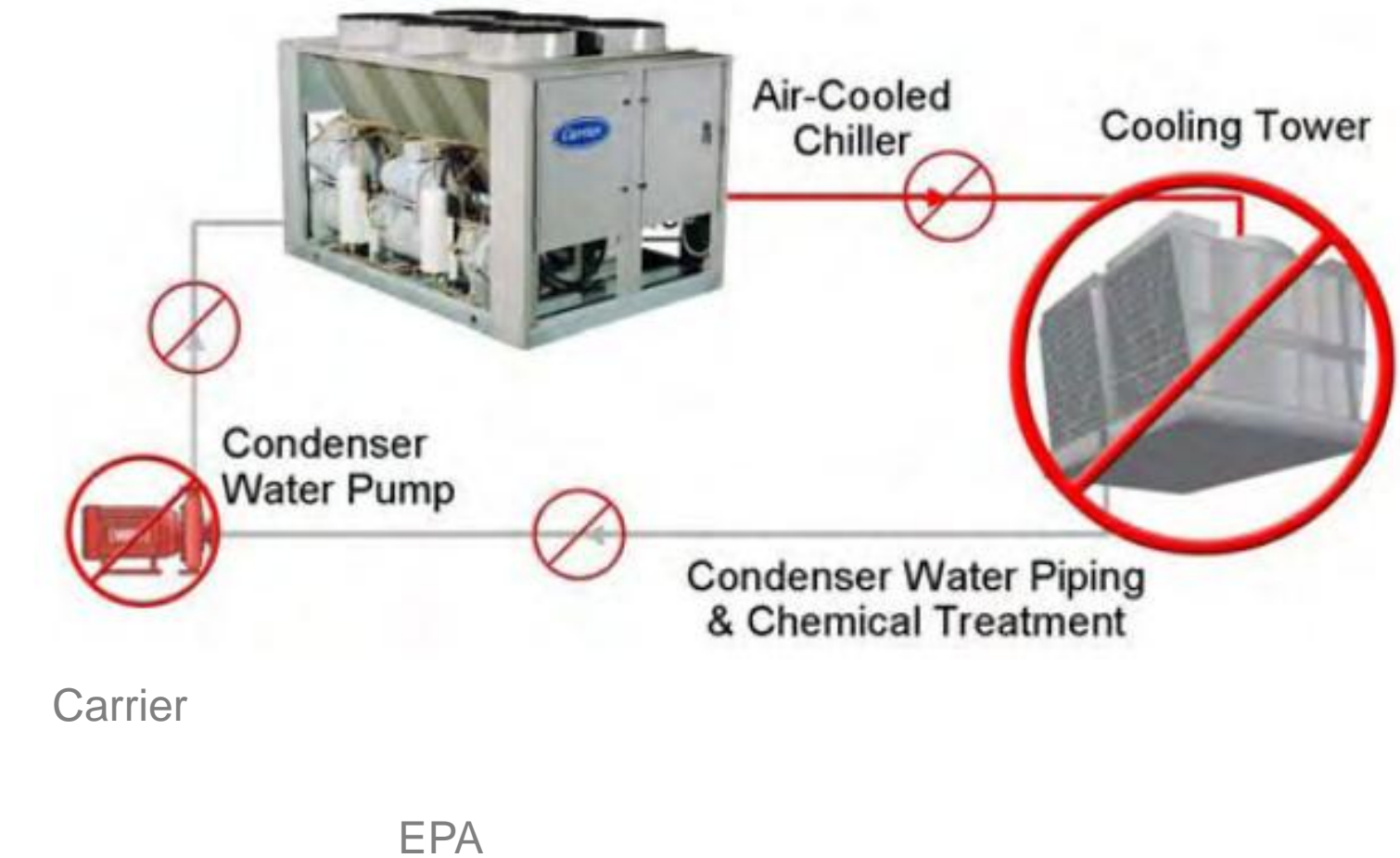
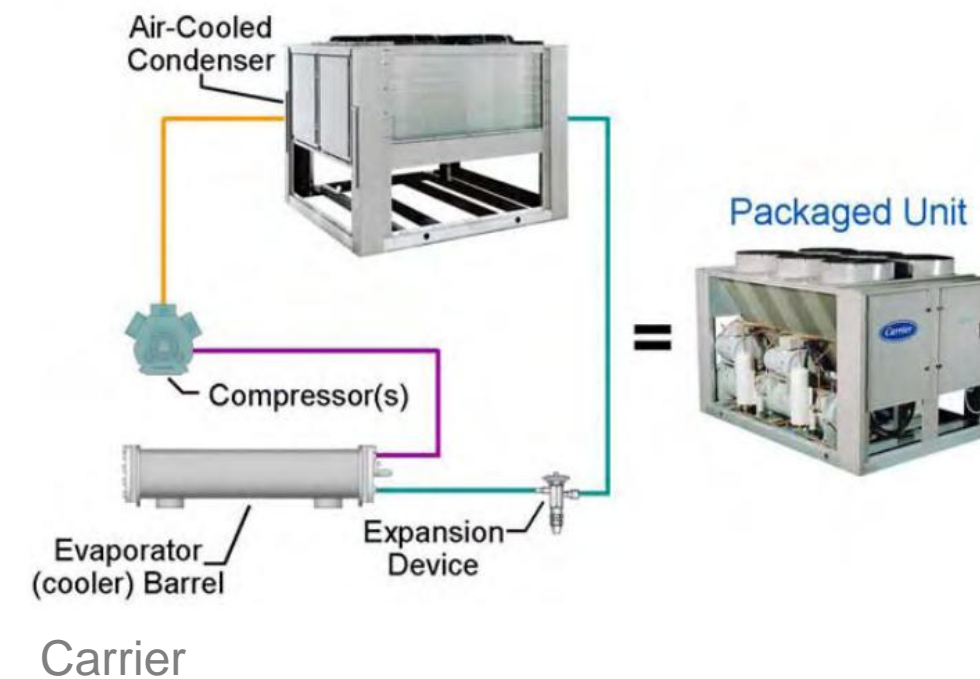
STRUCTURAL ANALYSIS



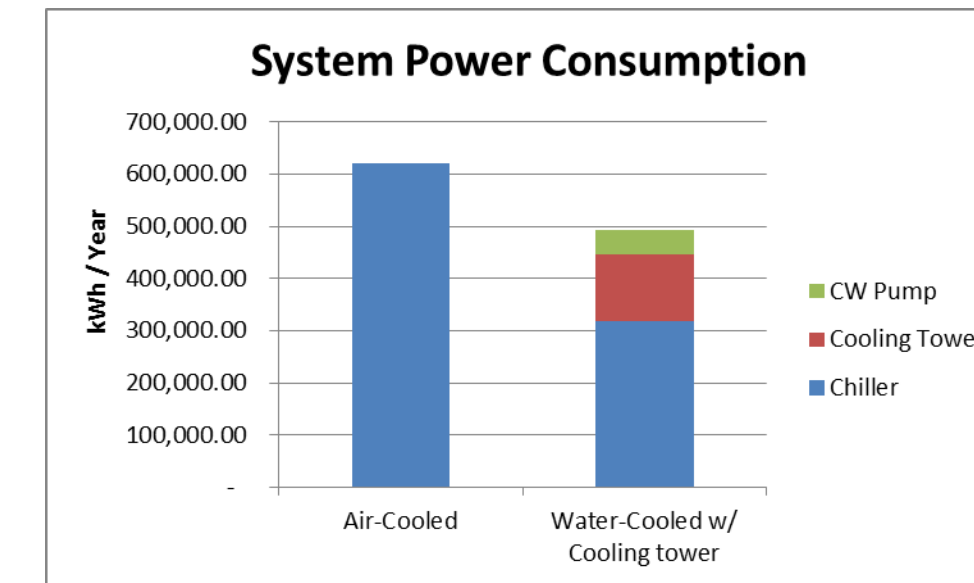
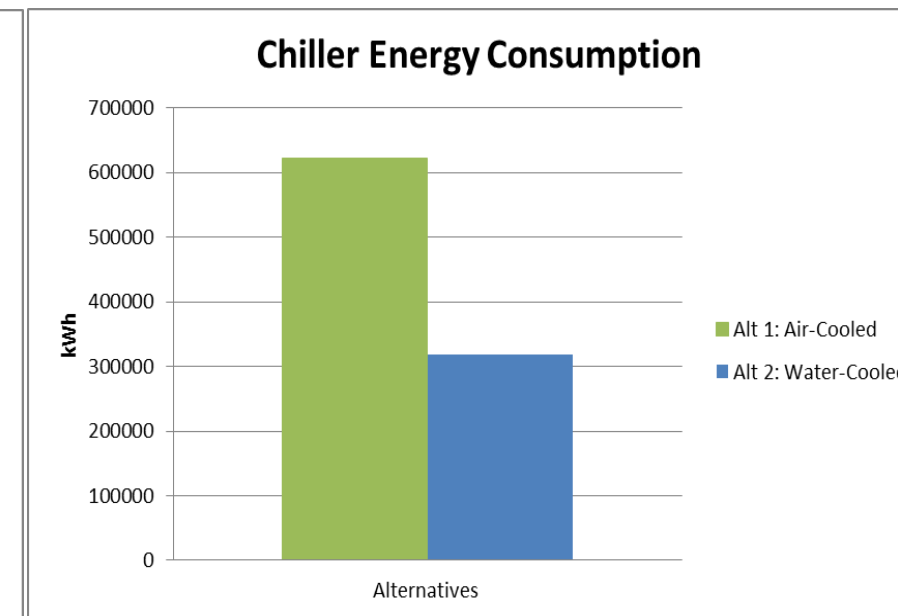
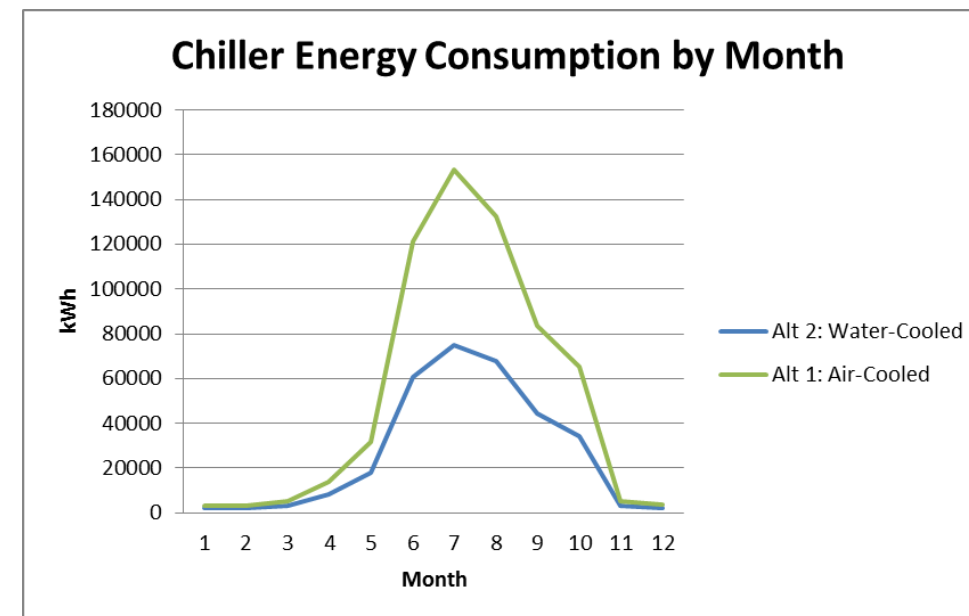
Member	Force	Allowable	Actual	OK?
Beam	Bending Moment	64.12 ft kips	18.5 ft kips	Yes
Beam	Shear	79.1 kips	3.7 kips	Yes
Girder	Bending Moment	3723 ft kips	686	Yes
Girder	Shear	886 kips	40.05 kips	Yes
Girder	Live Load Deflection	3.4"	2.75"	Yes
Girder	Total Deflection	6.85"	6.6"	Yes
Column	Load	355 kips	50	Yes
Thermal Break	Shear	13,400 psi	2.24 psi	Yes

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- **Mechanical Depth**
 - **Air vs. Water**
 - Cooling Tower vs. River Water
 - Conclusion
- Electrical Breadth
- Conclusion

Two 300 Ton Air Cooled Chillers EER 9.4



- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- **Mechanical Depth**
 - **Air vs. Water**
 - Cooling Tower vs. River Water
 - Conclusion
- Electrical Breadth
- Conclusion



Additional Water: 2,500 1000gal
 Cost : \$5,500.00 / year

CAPITAL			
Alternative 1: Air- Cooled		Alternative 2: Water-Cooled	
Equipment	Price	Equipment	Price
AC Chiller 1	\$215,000.00	WC Chiller 1	\$140,500.00
AC Chiller 2	\$215,000.00	WC Chiller 2	\$140,500.00
		Cooling Tower 1	\$37,000.00
		Cooling Tower 2	\$37,000.00
		CW Pump 1	\$5,575.00
		CW Pump 2	\$5,575.00
		CW Piping	\$21,000.00
4 Boilers	\$120,000.00	4 Boilers	\$120,000.00
Total	\$550,000.00	Total	\$507,150.00

Capital: -\$42,850

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- **Mechanical Depth**
 - **Air vs. Water**
 - Cooling Tower vs. River Water
 - Conclusion
- Electrical Breadth
- Conclusion

AIR VS. WATER

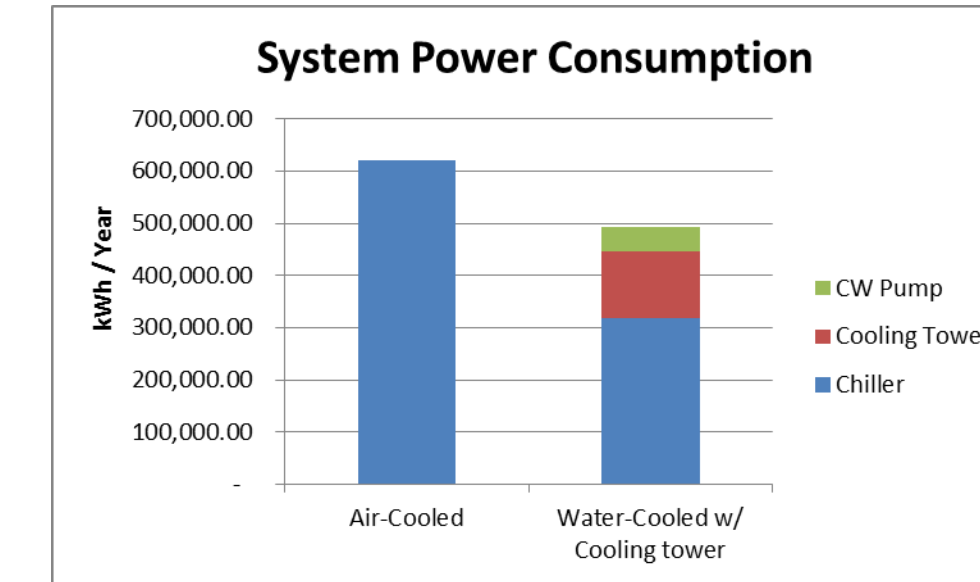
30 Year LCC

Existing Air-Cooled System: \$4,045,288.09

Alternative 2; Water-Cooled System: \$3,861,471.04

Savings: **\$183,817.05**

	*Total CO2e (lb):
Air-Cooled	9.01E+06
Water-Cooled	8.77E+06
% Diff	3%

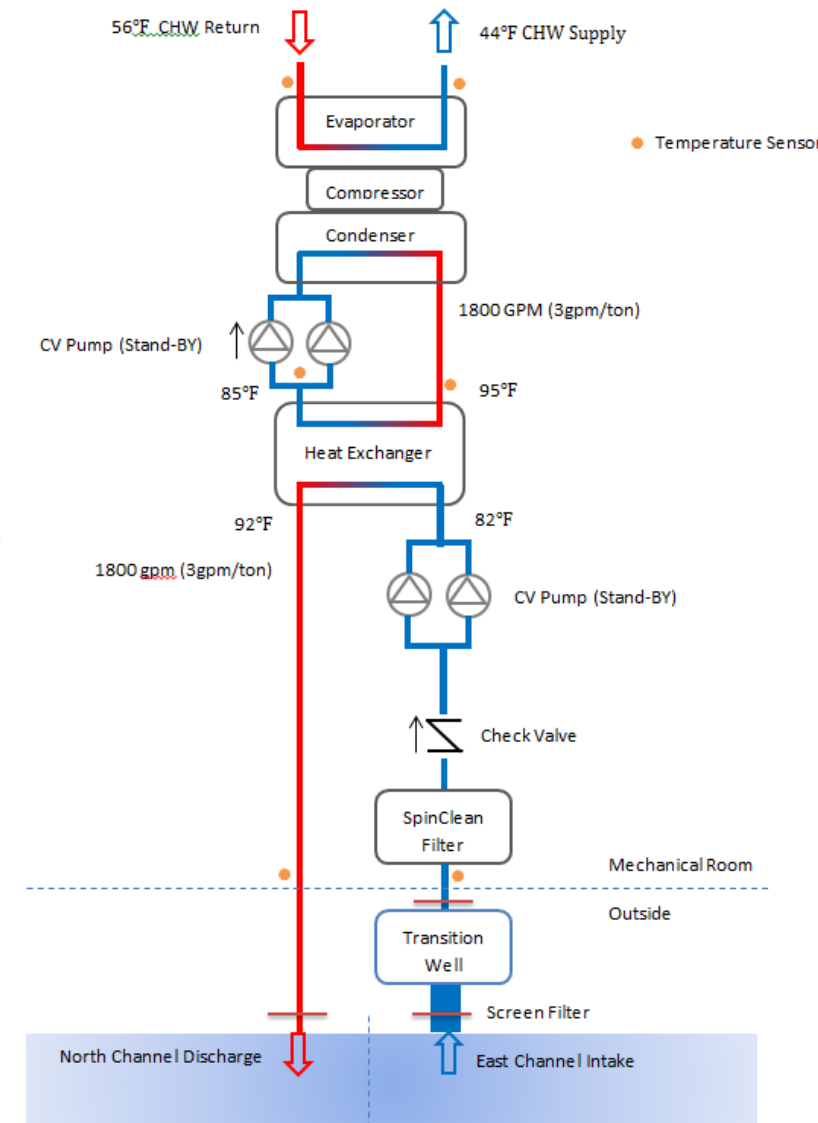


CAPITAL			
Alternative 1: Air- Cooled		Alternative 2: Water-Cooled	
Equipment	Price	Equipment	Price
AC Chiller 1	\$215,000.00	WC Chiller 1	\$140,500.00
AC Chiller 2	\$215,000.00	WC Chiller 2	\$140,500.00
		Cooling Tower 1	\$ 37,000.00
		Cooling Tower 2	\$ 37,000.00
		CW Pump 1	\$ 5,575.00
		CW Pump 2	\$ 5,575.00
		CW Piping	\$ 21,000.00
4 Boilers	\$120,000.00	4 Boilers	\$120,000.00
Total	\$550,000.00	Total	\$507,150.00

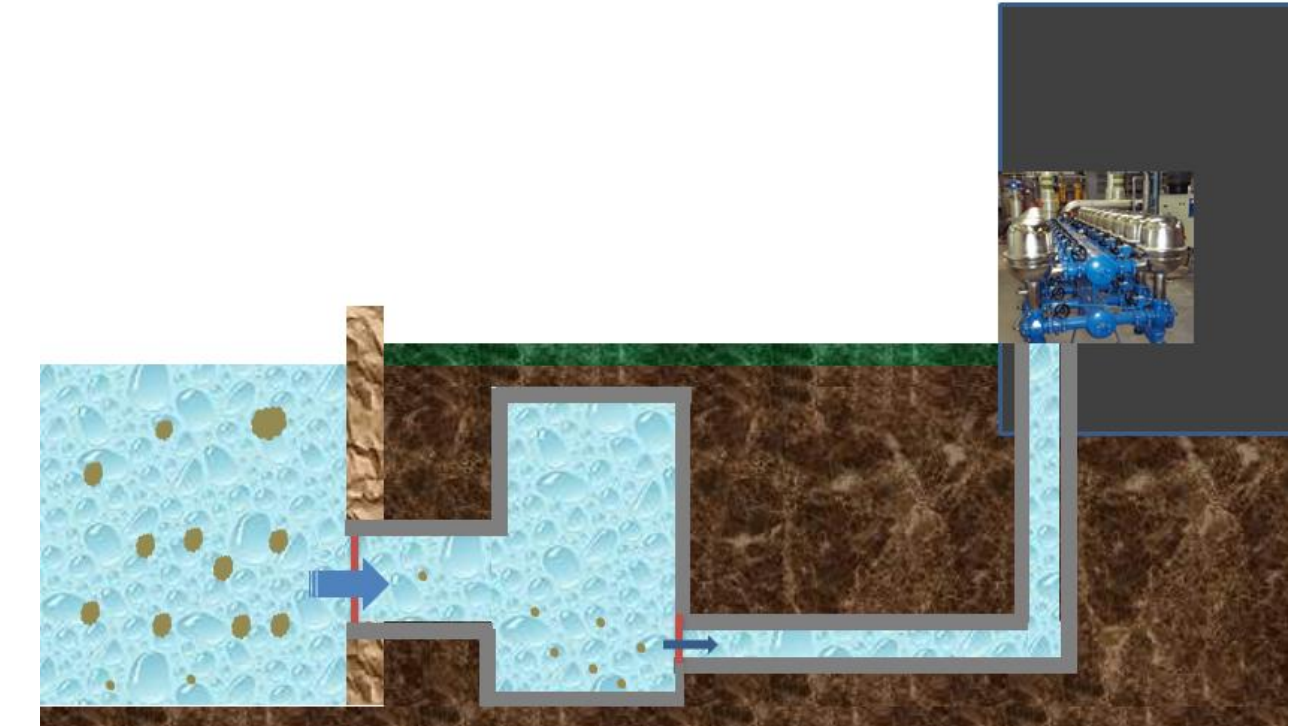
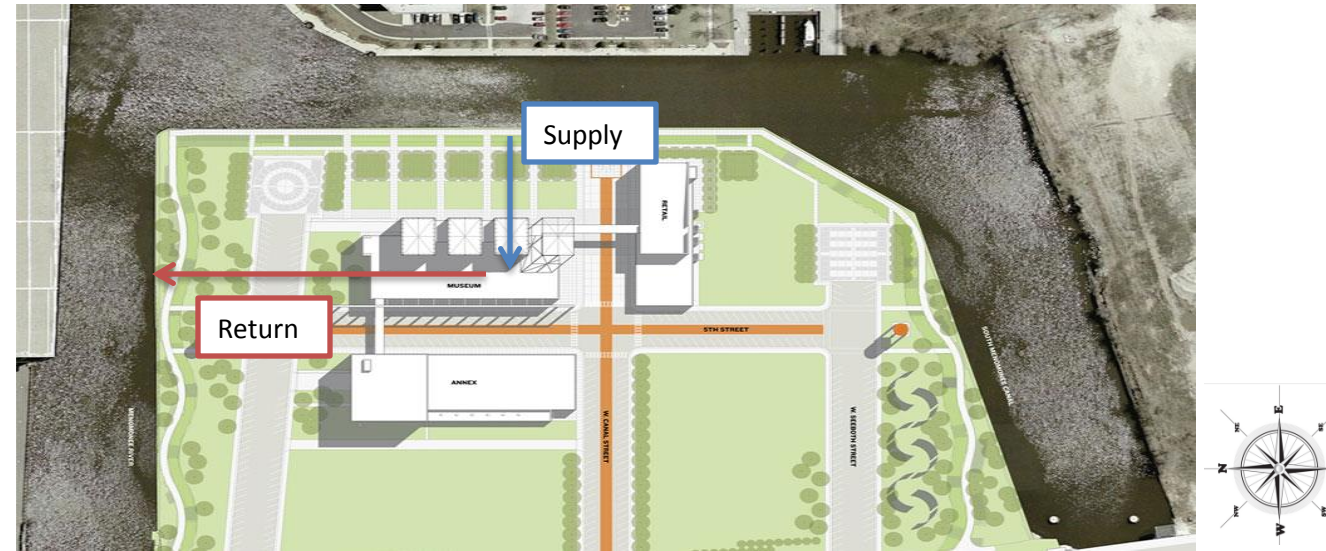
Capital: -\$42,850

PROJECT DEPTH

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- **Mechanical Depth**
 - Air vs. Water
 - **Cooling Tower vs. River Water**
- Conclusion
- Electrical Breadth
- Conclusion



COOLING TOWER VS. RIVER WATER

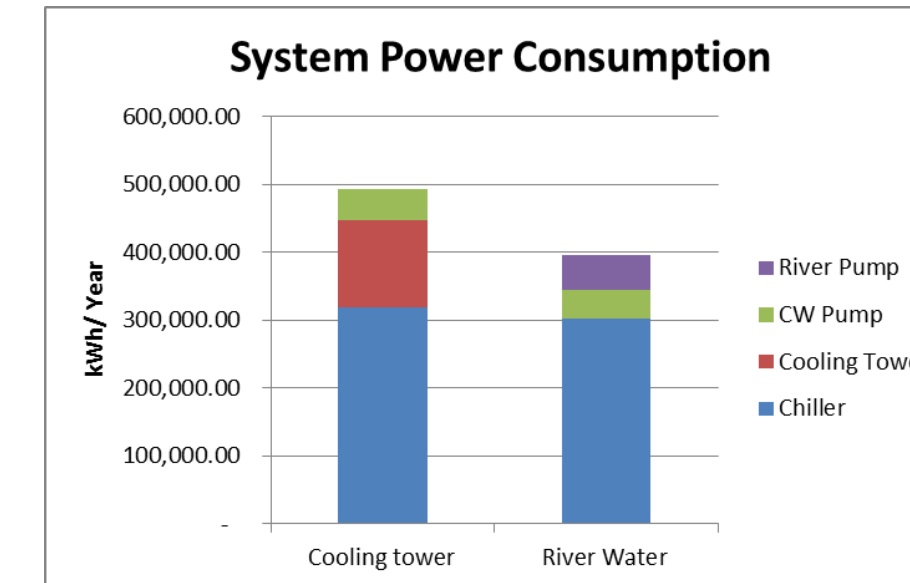
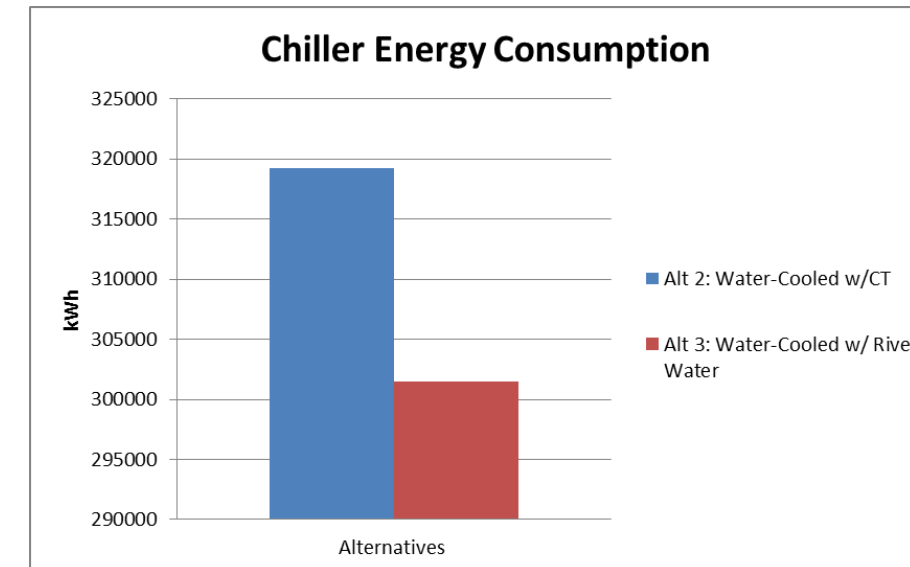
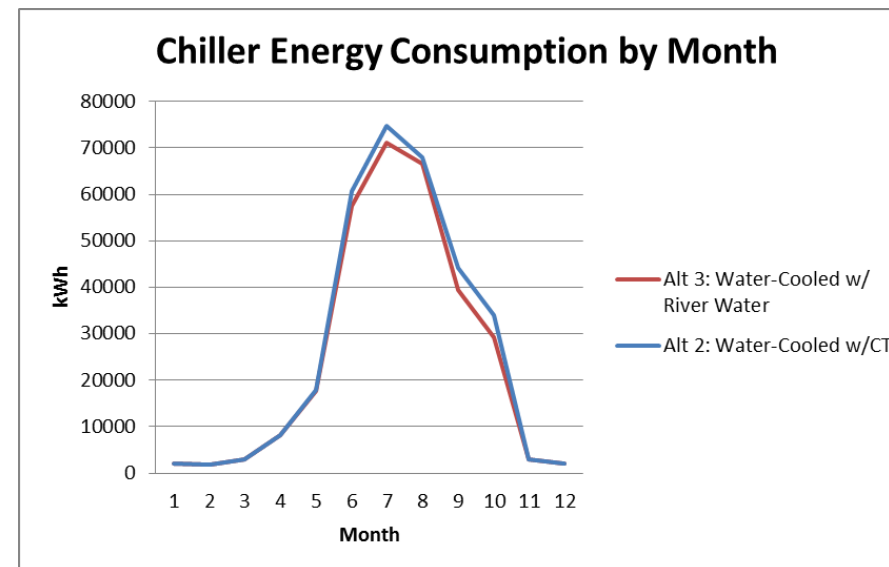


$$v = 0.4085 \frac{q}{d^2}$$

$v = \text{velocity} \left(\frac{ft}{s} \right)$, $q = \text{volume flow rate} \left(US \frac{gal}{min} \right)$,
 $d = \text{pipe inside diameter (inches)}$

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- **Mechanical Depth**
 - Air vs. Water
 - **Cooling Tower vs. River Water**
- Conclusion
- Electrical Breadth
- Conclusion

COOLING TOWER VS. RIVER WATER



CAPITAL			
Alternative 2: Water-Cooled Cooling Tower		Alternative 3: Water-Cooled River Water	
Equipment	Price	Equipment	Price
WC Chiller 1	\$ 140,500.00	WC Chiller 1	\$ 140,500.00
WC Chiller 2	\$ 140,500.00	WC Chiller 2	\$ 140,500.00
		River Pump 1	\$ 12,000.00
Cooling Tower 1	\$ 37,000.00	River Pump 2	\$ 12,000.00
Cooling Tower 2	\$ 37,000.00	River Piping	\$ 52,500.00
		Heat Exchanger	\$ 18,000.00
		Filtration System	\$ 100,000.00
CW Pump 1	\$ 5,575.00	CW Pump 1	\$ 5,575.00
CW Pump 2	\$ 5,575.00	CW Pump 2	\$ 5,575.00
CW Piping	\$ 20,995.00	CW Piping	\$ 4,750.00
4 Boilers	\$ 120,000.00	4 Boilers	\$ 120,000.00
Total	\$ 507,145.00		\$ 611,400.00

Capital: +\$104,255

PROJECT DEPTH

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- **Mechanical Depth**
 - Air vs. Water
 - **Cooling Tower vs. River Water**
- Conclusion
- Electrical Breadth
- Conclusion

COOLING TOWER VS. RIVER WATER

Simple Payback: **2.8 years**
 Discount payback : **3.0 years**

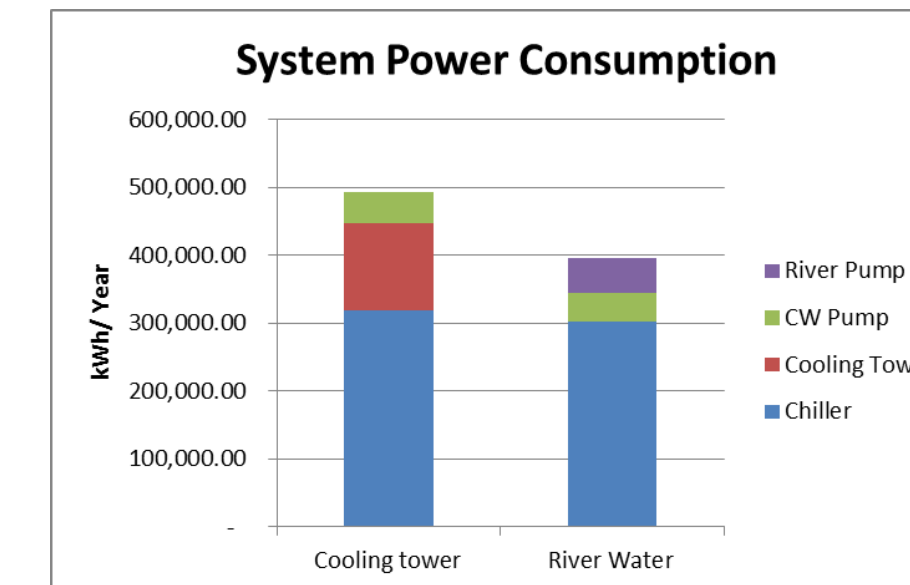
30 Year LCC

Alternative 2; Cooling Tower System: \$3,861,471.04

Alternative 3; River Water System: \$3,641,264.61

Savings: **\$220,206.43**

	*Total CO2e (lb):
Alt 2: Cooling Tower	8.77E+06
Alt 3: River Water	8.57E+06
% Diff	2%



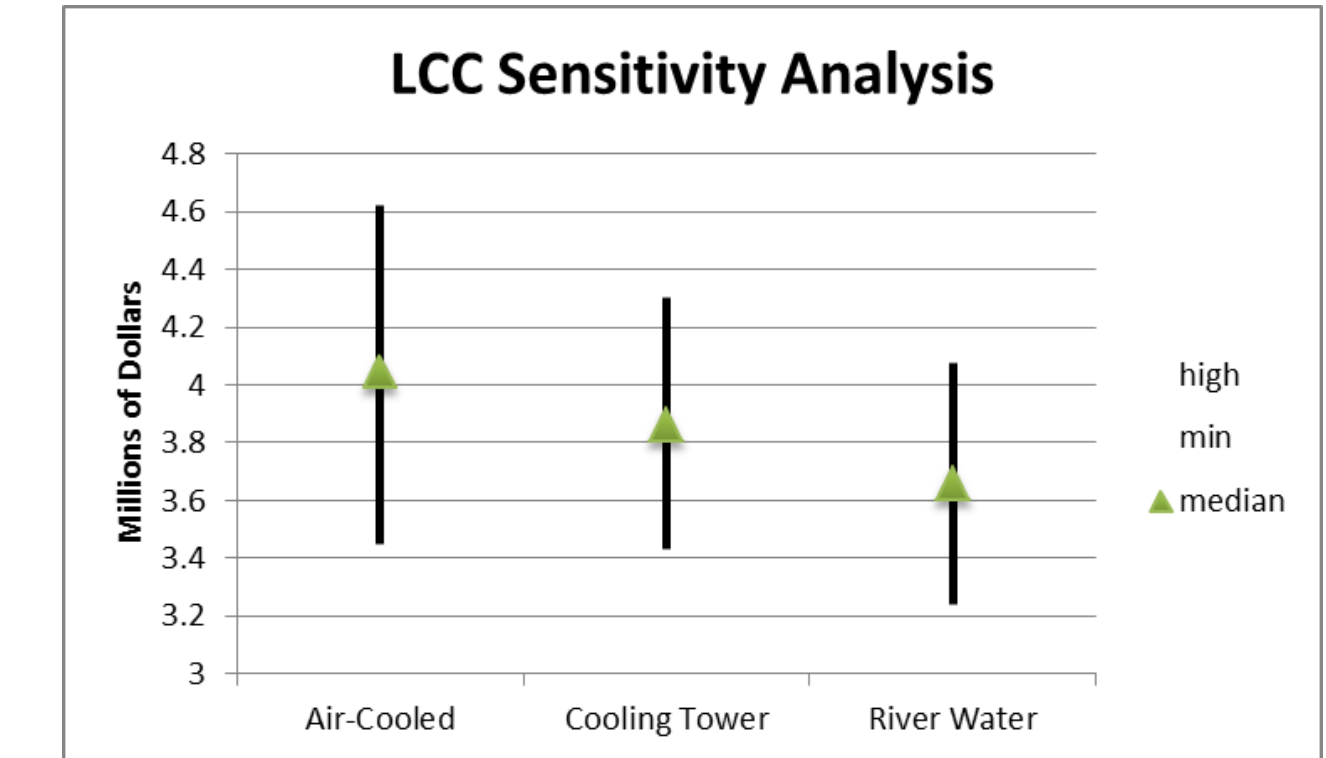
CAPITAL			
Alternative 2: Water-Cooled Cooling Tower		Alternative 3: Water-Cooled River Water	
Equipment	Price	Equipment	Price
WC Chiller 1	\$ 140,500.00	WC Chiller 1	\$ 140,500.00
WC Chiller 2	\$ 140,500.00	WC Chiller 2	\$ 140,500.00
		River Pump 1	\$ 12,000.00
Cooling Tower 1	\$ 37,000.00	River Pump 2	\$ 12,000.00
Cooling Tower 2	\$ 37,000.00	River Piping	\$ 52,500.00
		Heat Exchanger	\$ 18,000.00
		Filtration System	\$ 100,000.00
CW Pump 1	\$ 5,575.00	CW Pump 1	\$ 5,575.00
CW Pump 2	\$ 5,575.00	CW Pump 2	\$ 5,575.00
CW Piping	\$ 20,995.00	CW Piping	\$ 4,750.00
4 Boilers	\$ 120,000.00	4 Boilers	\$ 120,000.00
Total	\$ 507,145.00		\$ 611,400.00

Capital: +\$104,255

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- **Mechanical Depth**
 - Air vs. Water
 - Cooling Tower vs. River Water
 - **Conclusion**
- Electrical Breadth
- Conclusion

Air-Cooled vs. Water-Cooled with River Water

- Capital cost increased 10% [\$61,400.00]
- Annual operating cost reduced by 14% [\$21,732.00]
- 30 year LCC reduced 10% [\$389,986.00]
- Simple payback 3 years



ELECTRICAL BREADTH

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- Mechanical Depth
- **Electrical Breadth**
 - **CHP Feasibility**
 - Conclusion
- Conclusion

CHP FEASIBILITY

Spark Gap

Electric Cost: \$0.10/kWh = \$29.30/MMBTU

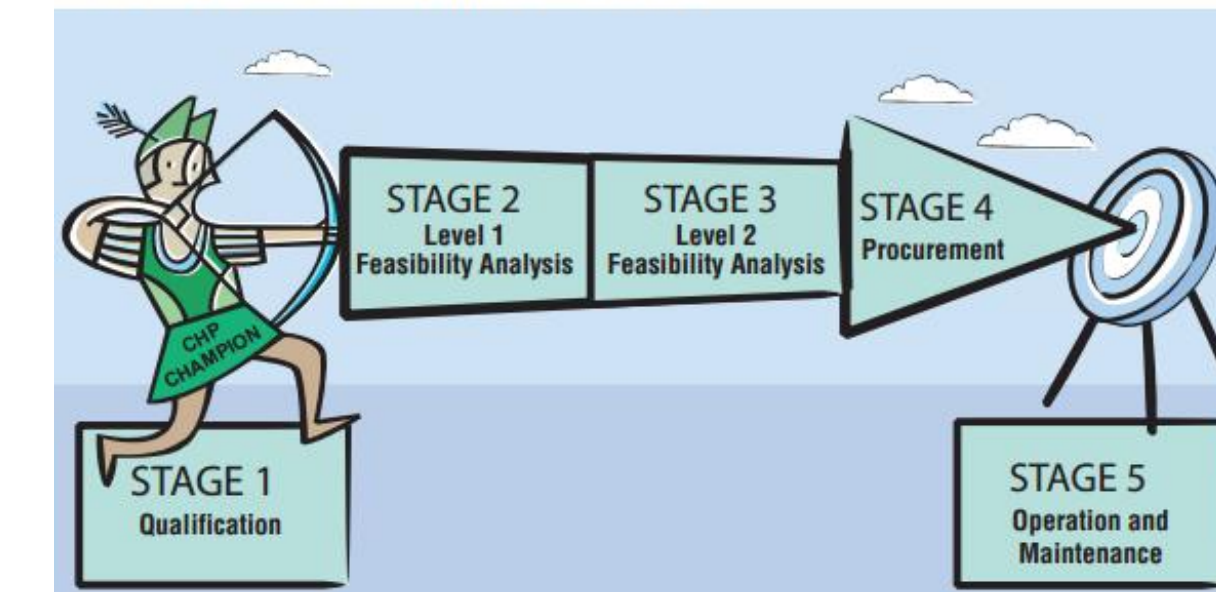
Gas Cost: : \$0.80/therm = \$8.00/MMBTU

1 : 3.7 Ratio , 1 : 4 [rule of thumb]



What You Need to Know

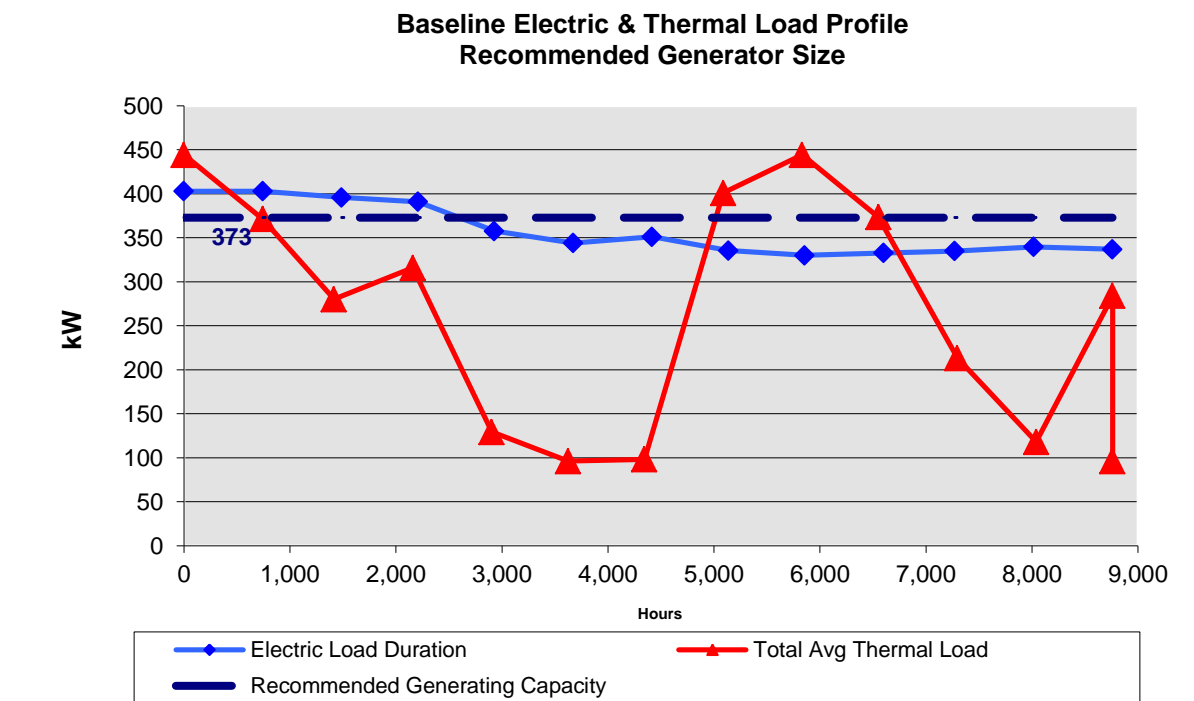
CHP Project Development Process



- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- Mechanical Depth
- **Electrical Breadth**
 - **CHP Feasibility**
 - Conclusion
- Conclusion

Thermal-to-Electric Ratio = 0.74

Recommended: 373 kWe Gas Engine



- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- Mechanical Depth
- **Electrical Breadth**
 - CHP Feasibility
 - **Conclusion**
- Conclusion

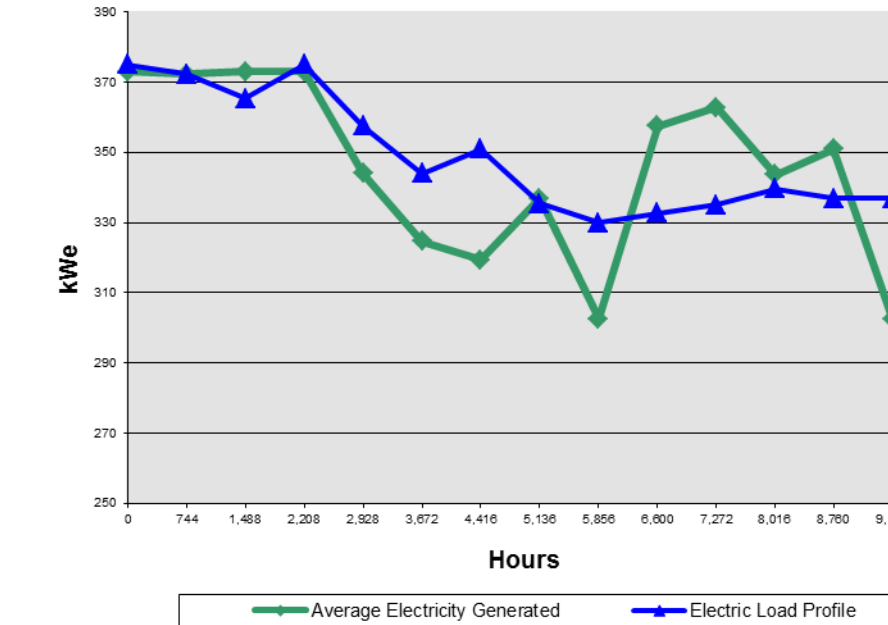
CHP CONCLUSION

- Additional Cost \$564,000
- Generation Cost \$0.088 /kWh
1.2 cents less than purchased
- Total Savings: \$140,000 /year
- Simple Payback: 4.04 Years
- CO₂ reduction of 62%

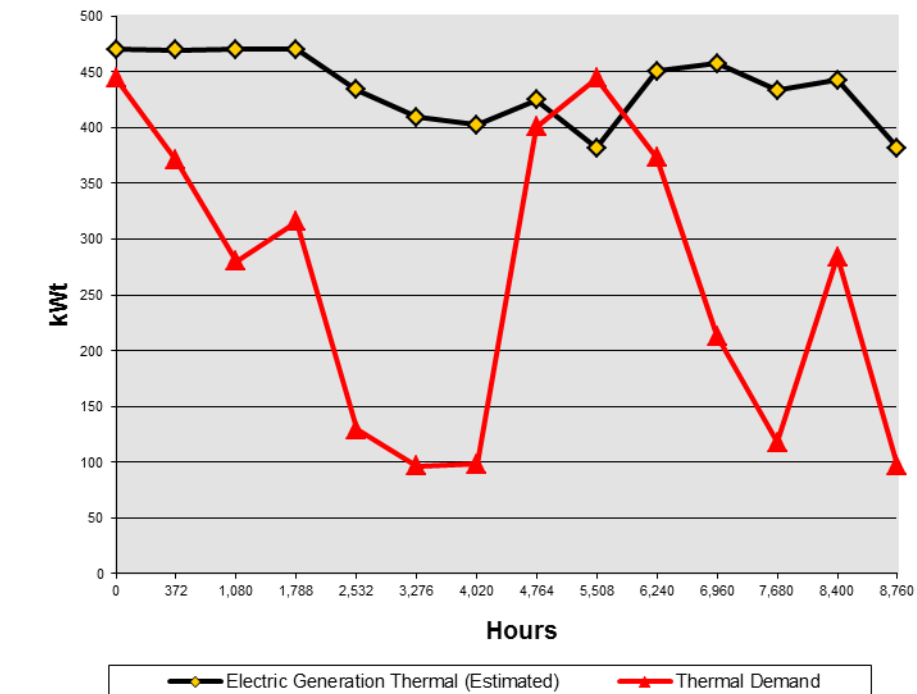
CHP Results



Comparison of Generation to CHP Electric Load

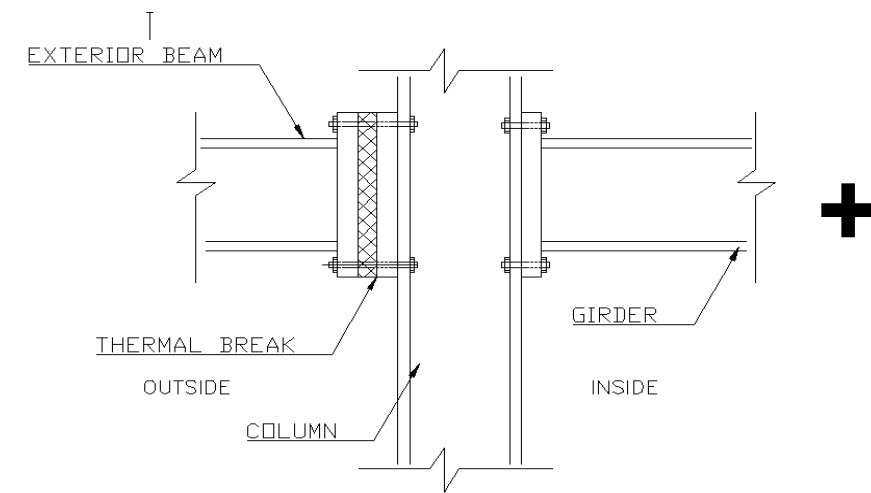


Thermal Demand vs CHP Generation Thermal

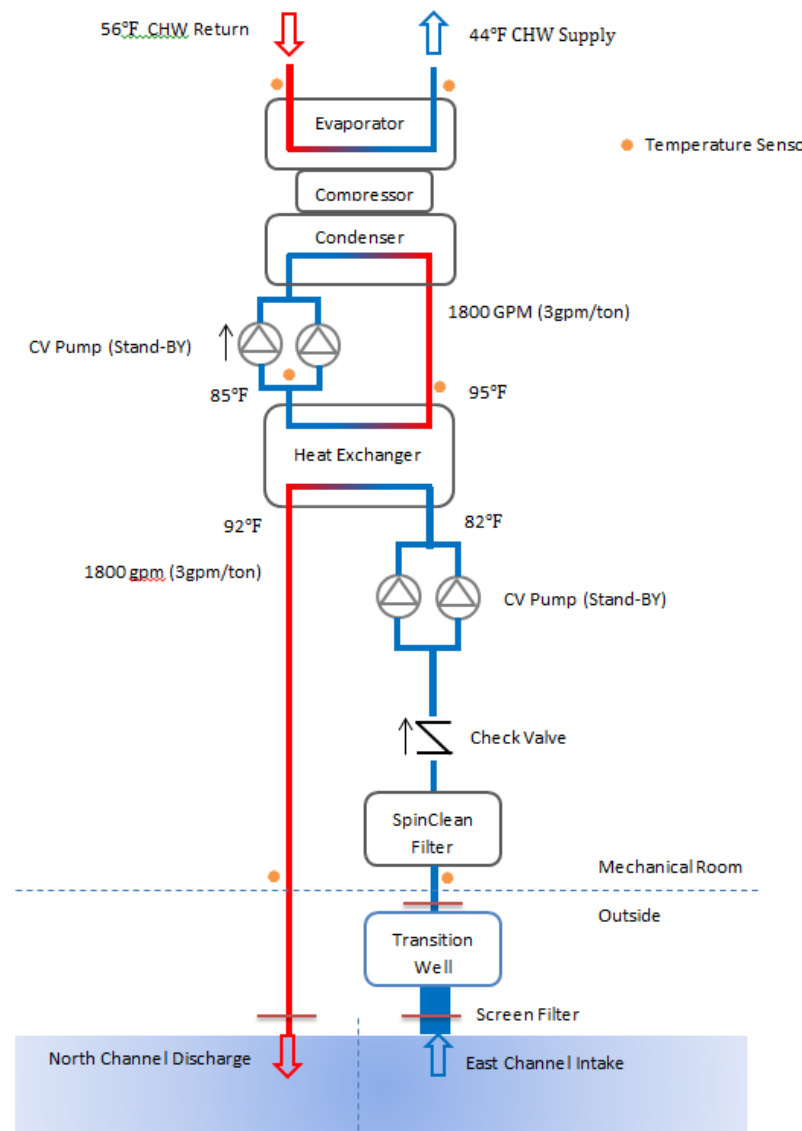


CONCLUSION

- Project Background
- Thesis Goals
- Existing Conditions
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- **Conclusion**



+



+



=

- Total**
- \$160,000 Annually
 - 4 Year Payback
 - 65% Reduction of CO₂



- Project Background
- Existing Conditions
- Thesis Goals
- Structural Breadth
- Mechanical Depth
- Electrical Breadth
- **Conclusion**



THANK YOU

HGA

Kevin Pope, P.E.
Jeff Harris, P.E.

Steve Mettlach

Harley-Davidson

Joyce Koker, P.E.

Penn State

Dr. William Bahnfleth
Dr. Jelena Srebric
Dr. James Freihaut
Mr. David H. Tran

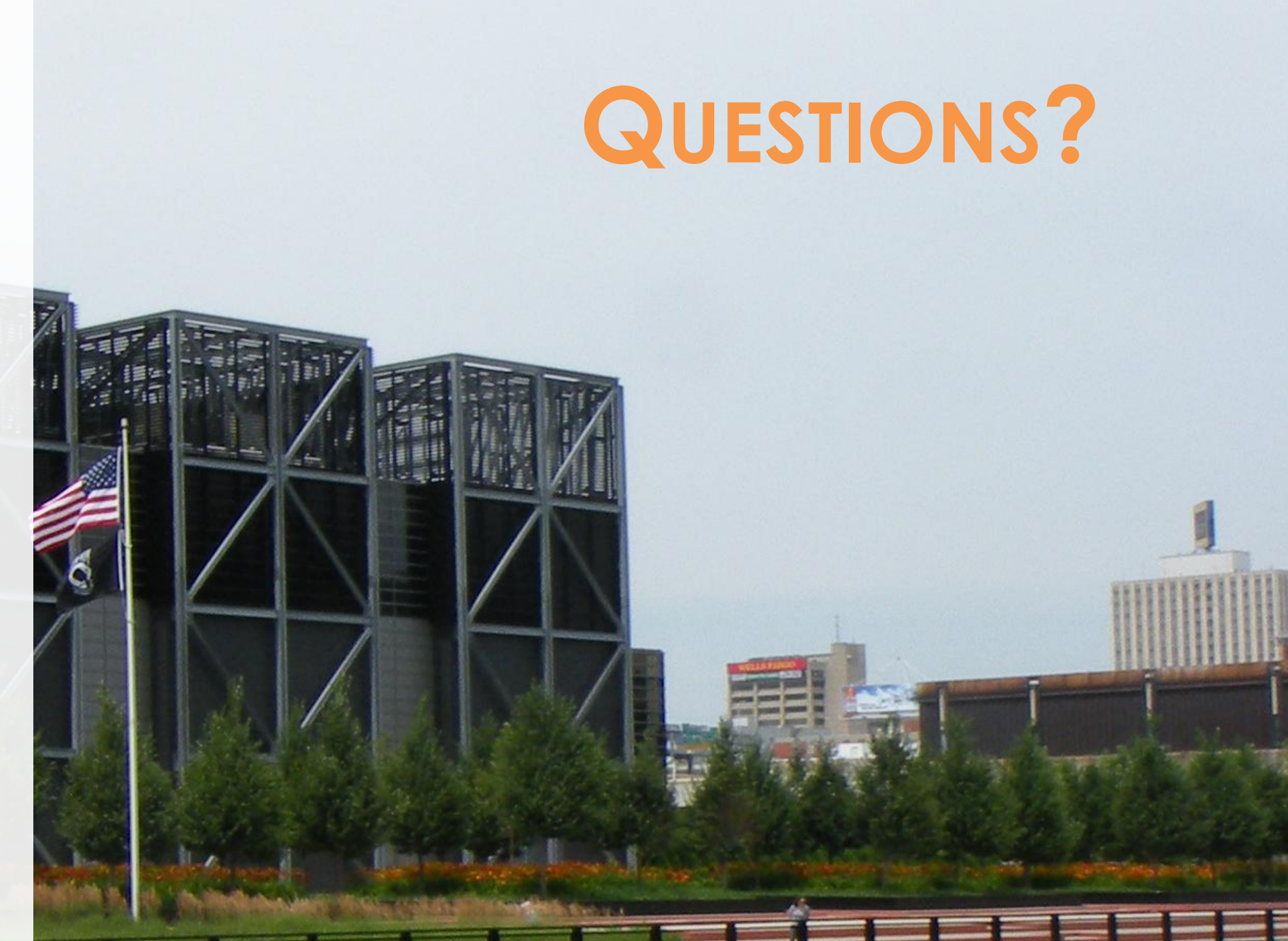
Family and friends for their support

Associate Vice President, HGA
Director of Mechanical Engineering,
HGA, Penn State Alumni
Mechanical Engineer, HGA

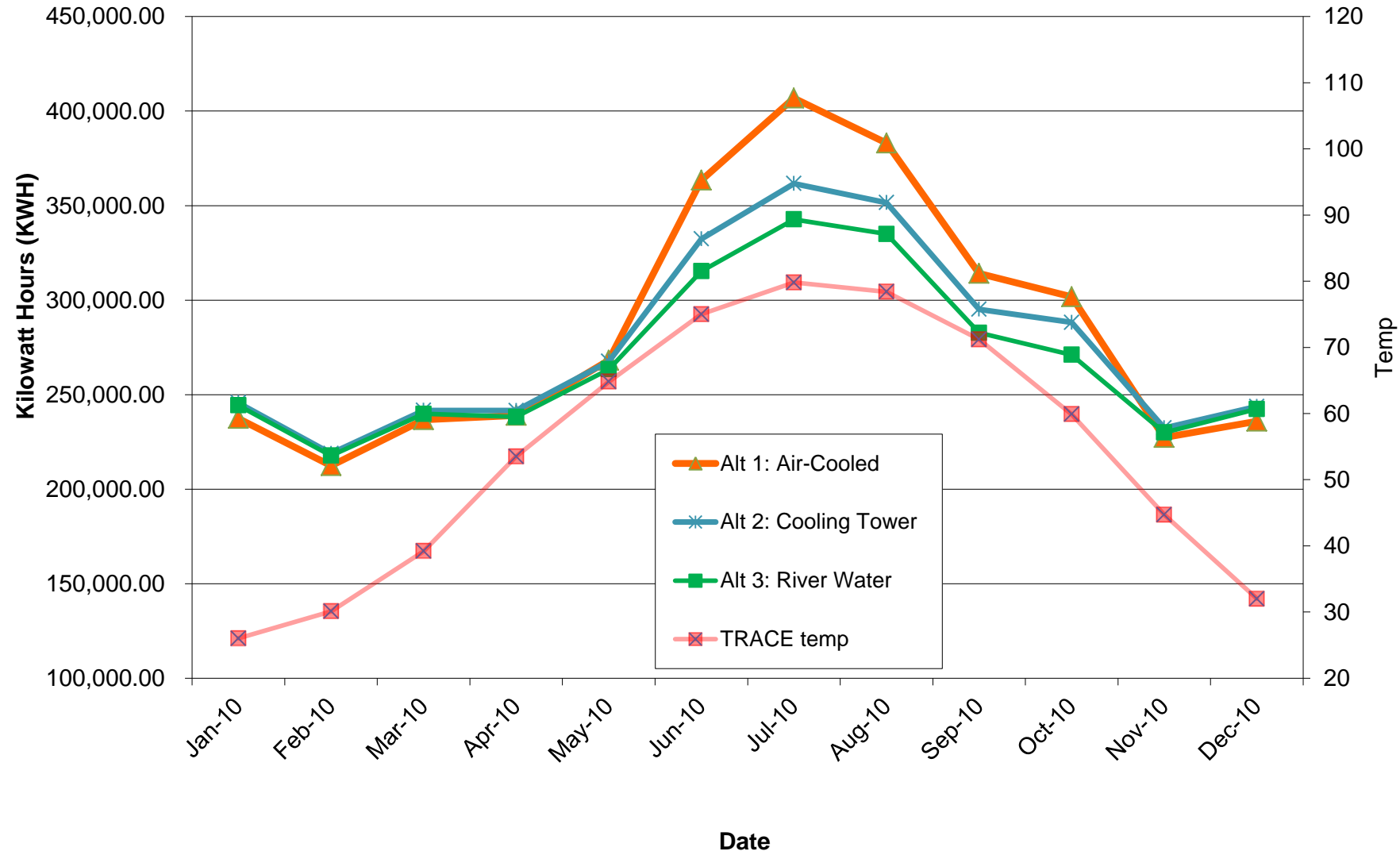
Harley-Davidson Museum

Faculty Advisor
AE Mechanical Professor
AE Mechanical Associate Professor
AE 5th Year Structural, BAE/MS

QUESTIONS?



Museum Campus Electricity Use



Air-Cooled			
	Elec kWh	Water 1000gal	Gas therms
Elec	2,168,082.30		
Air Side	569,425.50		
Water Side	683,862.00		
Hot water	12,833.20	74.90	37,736.30
total	3,434,203.00	74.90	37,736.30

Water-Cooled with Cooling Tower			
	Elec kWh	Water 1000gal	Gas therms
Elec	2,168,082.30		
Air Side	569,425.50		
Water Side	557,153.20	2,505.10	
Hot water	12,833.20	74.90	37,736.30
total	3,307,494.20	2,580.00	37,736.30

Water Cooled with River			
	Elec kWh	Water 1000gal	Gas therms
Elec	2,168,082.30		
Air Side	569,425.50		
Water Side	459,295.80		
Hot water	12,833.20	74.90	37,736.30
total	3,209,636.80	74.90	37,736.30

	Alternative	LCC
Air-cooled	1	\$4,046,288.09
Cooling Tower	2	\$ 3,861,471.04
River	3	\$ 3,656,301.16

Alternative 1: Air-Cooled			
	Elec kWh	Water 1000gal	Gas therms
Air Side	569,425.50	-	-
Water Side	683,862.00	-	-
Hot water	12,833.20	74.90	37,736.30
total	1,266,120.70	74.90	37,736.30
Price per unit	\$ 0.10	\$ 2.20	\$ 0.80
Cost	\$ 126,612.07	\$ 164.78	\$ 30,189.04

Economic Life	30	years
Overhaul	\$ 15,000.00	every 7 years up tp 21
Maintenance	\$ 5,000.00	per year
Discount Rate	2.3%	DR

AirSide = AHUs
 Water Side = Chiller and CHW pump
 Hot Water = Boiler and HW pump

Equipment	Price
AC Chiller 1	\$ 215,000.00
AC Chiller 2	\$ 215,000.00
4 Boilers	\$ 120,000.00
Capital	\$ 550,000.00

Capital \$ 550,000.00

	Electricity			Fuels		
	Energy	Peak Demand	% of Total Cost from Peak Demand	Cost*	Gas	
					Energy	Cost*
	<i>kWh</i>	<i>kW</i>		\$	<i>Therms</i>	\$
Jan-03	252,717	611	10%	\$25,272	12,113	\$9,690
Feb-03	225,119	626	10%	\$22,512	8,256	\$6,605
Mar-03	247,492	631	10%	\$24,749	4,217	\$3,374
Apr-03	241,526	671	10%	\$24,153	3,102	\$2,482
May-02	255,900	704	10%	\$25,590	3,848	\$3,078
Jun-02	281,412	900	10%	\$28,141	11,795	\$9,436
Jul-02	299,775	932	10%	\$29,978	14,495	\$11,596
Aug-02	294,522	898	10%	\$29,452	13,080	\$10,464
Sep-02	257,444	847	10%	\$25,744	8,969	\$7,175
Oct-02	261,155	799	10%	\$26,116	6,960	\$5,568
Nov-02	237,578	631	10%	\$23,758	3,040	\$2,432
Dec-02	250,657	618	10%	\$25,066	10,303	\$8,242

Thermal-to-Electric Ratio	= 0.74
Recommended Prime Mover(s)	
Gas Engine	Recommended
Microturbine	
Gas Turbine (Simple Cycle)	
Phosphoric Acid Fuel Cells	

Select Prime Mover	Gas Engine
Would backup generation have been installed anyway?	Yes
If "Yes", indicated planned size	300 kWe
Recommended Generation	373 kWe
Chose Size (per Unit)	373 kWe
Chose Number of Units	1 Unit(s)
Total Selected Capacity	373 kWe
Electric Efficiency	34 %
Gross Heat Rate Exhaust (LHV)	6,623 BTU/kWe
Recoverable Heat Rate (LHV)	4,305 BTU/kWe
O&M Costs	\$0.0120 \$/kWh/yr
Electric Use	0.0000 kWe/kWe

Include Absorption Chiller	Yes
Will the Absorption Chiller Displace an Electric Chiller?	Yes
Size	89 RT
Thermal Input	1,606 MBTU/hr
O&M Costs	\$55 \$/RT/yr
Electric Use	0.0300 kWe/RT
Electric Displaced	0.6000 kWe/RT

Select Desiccant	None
Would a desiccant have been installed	No
Chose Size (per Unit)	10,000 SCFM
Chose Number of Units	1 Unit(s)
Total Selected Capacity	SCFM
Regeneration Requirements (200°F)	BTU/hr
O&M Costs	¢/SCFM/yr
Latent Heat Removal Rate	BTU/hr
Electric Use	kWe per kSCFM

SITE		RESULTS	
MN Hospital 1234 W. Main St Milwaukee, WI			
ASSUMPTIONS		CHP RESULTS	
Average Electric Cost	10.000 ¢/kWh	Prime Mover	
Peak Average Electric Cost	N/A ¢/kWh	Total ECP Cost	\$564 \$(1000)
Initial Electric Sell Back	1.500 ¢/kWh	Prime Mover	Gas Engine
Supplemental Elect Cost	10.000 ¢/kWh	Parasitic Load	2.7 kW
Cogen Initial Fuel Cost	8.00 \$/MMBTU	Total Generation Capacity	373 kW
W/O Cogen Fuel Cost	8.00 \$/MMBTU	Electrical Output	3,035 MWh
Existing Boiler Efficiency	86.0 %	Absorption Chiller Credit	60 MWh
Standby Demand Charge	\$1.50 \$/kw/month	Net Total Generation Effect	3,095 MWh
Standby Capacity Required	373 kW	Eleelectric Capacity Factor	93 %
O&M Charge	\$4,943 \$/yr	Gross Heat Rate (LHV)	10,038 BTU/kWh
Annual Electric Load	3,105 MWh	Recoverable Heat	4,305 BTU/kWh
Annual Heat Load	7,795 MMBTU	Thermal Loads	
		TAT Thermal Loads (June, July, August)	
PURPA	(Assuming Gas or Liquid Fuel Fired)	Absorption Chiller	1,795 MMBTU
Efficiency	55.4 %	Desiccant	0 MMBTU
Qualified Facility	Yes	Total Thermal Load with TAT	9,590 MMBTU
Sell Back	0 kWh	Thermal Capacity Factor	78 %
Sell Back Desired	No	Thermal Energy Output	
		From Generator	13,065 MMBTU
		From Auxiliary Boiler	0 MMBTU
FINANCIAL RESULTS			
COSTS WITHOUT COGENERATION \$(1000)		Fuel Requirements:	
Electricity Costs	\$311	For Generator (HHV)	33,669 MMBTU
Thermal Energy Costs	\$80	For Auxiliary Boiler (HHV)	0 MMBTU
	TOTAL		
	\$391		
COSTS WITH COGENERATION \$(1000)		Generation Costs	8.88 ¢/kWh
Supplemental Electric Purchase	\$1		
Peak Electric Charge Adjustment	(\$31)		
Fuel	\$269		
Electricity Sold	\$0		
O&M	\$5		
Standby Charges	\$7		
	TOTAL		
	\$251		
	SAVINGS		\$140
	SIMPLE PAYBACK	4.04	Years

